

A Micromechanics Finite Element Model for Studying the Mechanical Behavior of Spray-On Foam Insulation (SOFI)

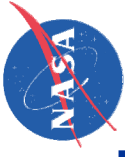
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Proposed Abstract for the
43rd Annual Technical Meeting
Society of Engineering Science
August 13-16th, 2006

A micromechanics model has been constructed to study the mechanical behavior of spray-on foam insulation (SOFI) for the external tank. The model was constructed using finite elements representing the fundamental repeating unit of the SOFI microstructure. The details of the micromechanics model were based on cell observations and measured average cell dimensions discerned from photomicrographs. The unit cell model is an elongated Kelvin model (fourteen-sided polyhedron with 8 hexagonal and six quadrilateral faces), which will pack to a 100% density. The cell faces and cell edges are modeled using three-dimensional 20-node brick elements. Only one-eighth of the cell is modeled due to symmetry.

By exercising the model and correlating the results with the macro-mechanical foam behavior obtained through material characterization testing, the intrinsic stiffness and Poisson's Ratio of the polymeric cell walls and edges are determined as a function of temperature. The model is then exercised to study the unique and complex temperature-dependent mechanical behavior as well as the fracture initiation and propagation at the microscopic unit cell level.



A Micromechanics Finite Element Model for Studying the Mechanical Behavior of Spray-On Foam Insulation

Louis J. Ghosn

*Senior Researcher
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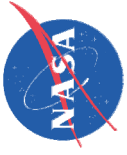
Roy M. Sullivan

*Material Research Engineer
NASA Glenn Research Center*

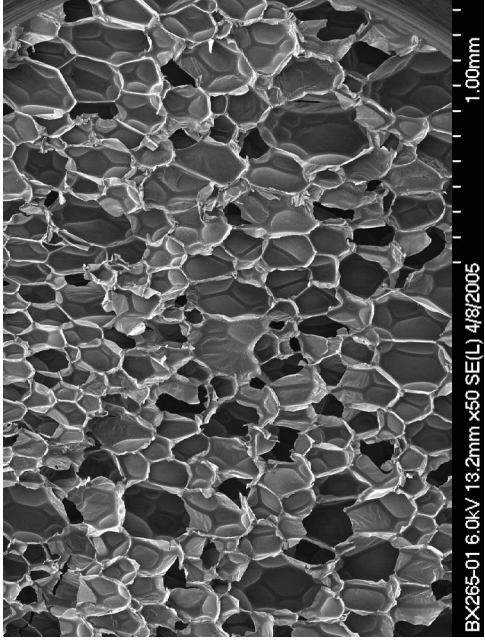
Bradley A. Lerch

*Principal Scientist
NASA Glenn Research Center*

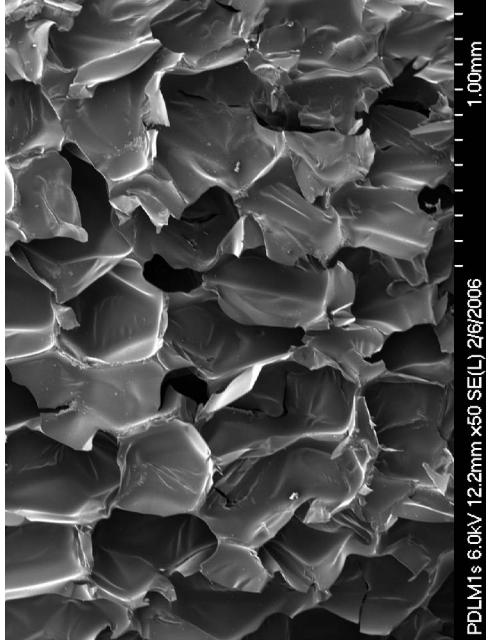
43rd Annual Technical Meeting
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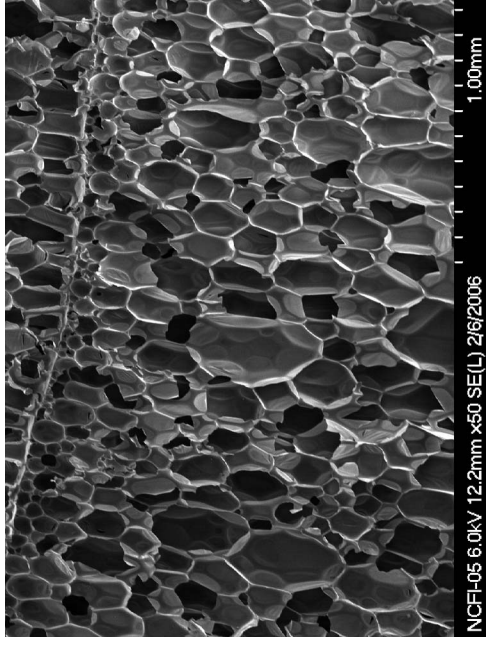
Micrographs of the Three Major NASA's SOFI Materials



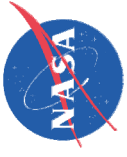
BX-265



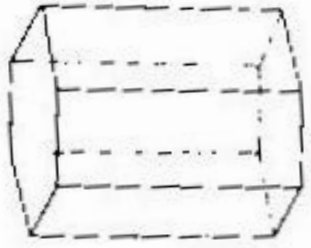
PDL



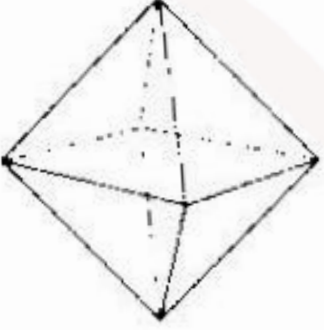
NCFI



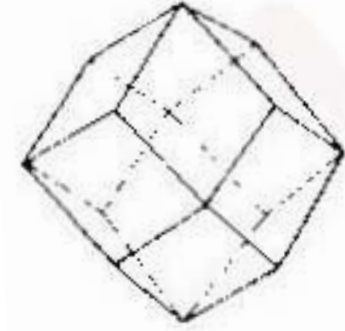
Classical Three-Dimensional Polyhedral Cells



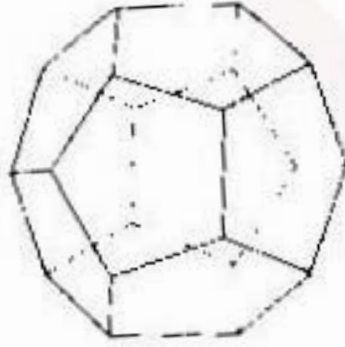
Hexagonal Prism
8 faces, 18 edges
Packs to fill volume ✓



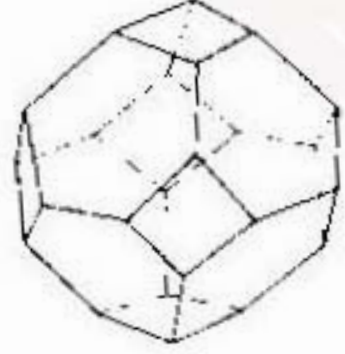
Octahedron
8 faces, 12 edges



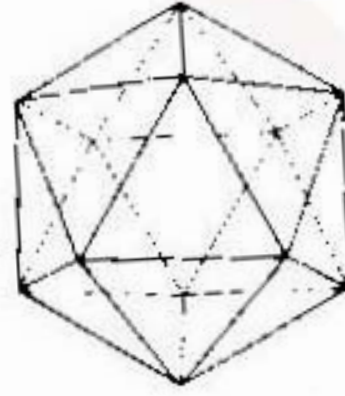
Rhombic dodecahedron
12 faces, 24 edges
Packs to fill volume ✓



Pentagonal Dodecahedron
12 faces, 30 edges

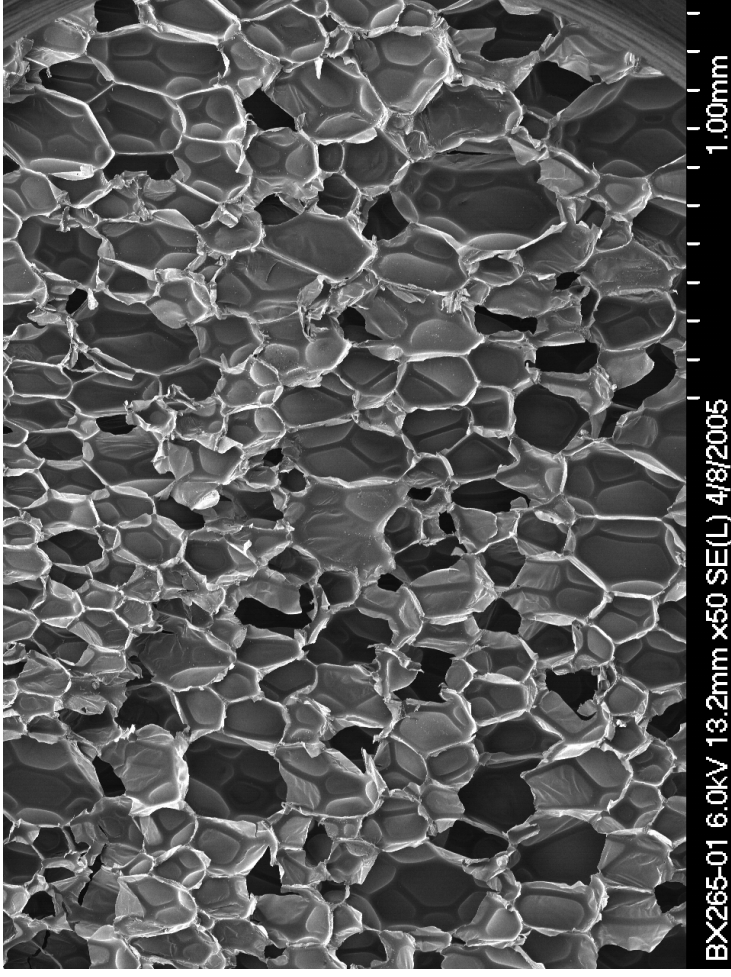


Tetraikaidecahedron
14 faces, 36 edges
Packs to fill volume ✓



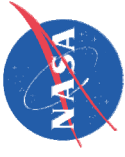
Icosahedron
20 faces, 30 edges

BX-265

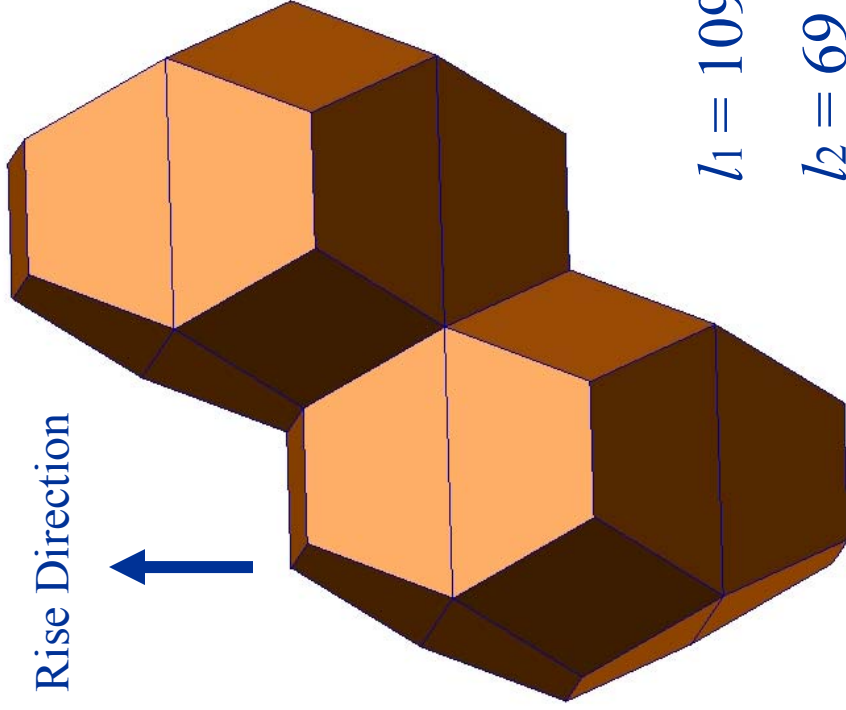


Rise Direction

- Average Cell Length (parallel to rise) – 218 μm
- Average Cell Width (perpendicular to rise) – 124 μm
 - Average measured # of faces = 12.4
 - Observed elongated cell structure
- Cells consist of thick edges and thin faces



Assumed Elongated Tetraikadechaedron BX-265



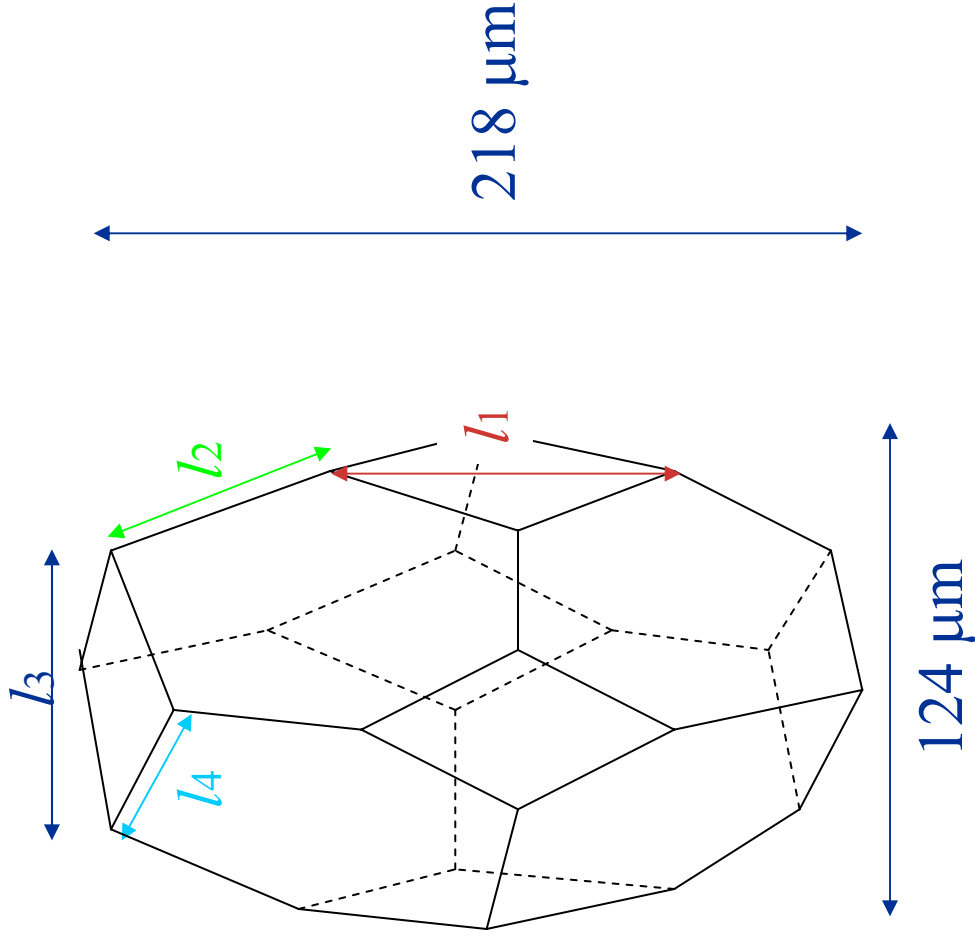
$$l_1 = 109 \mu\text{m}$$

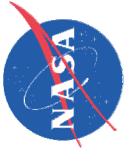
$$l_2 = 69 \mu\text{m}$$

$$l_3 = 39 \mu\text{m}$$

$$l_4 = 28 \mu\text{m}$$

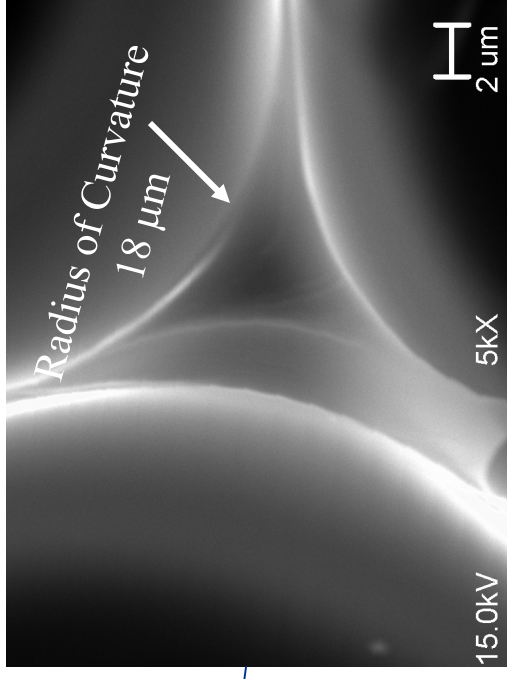
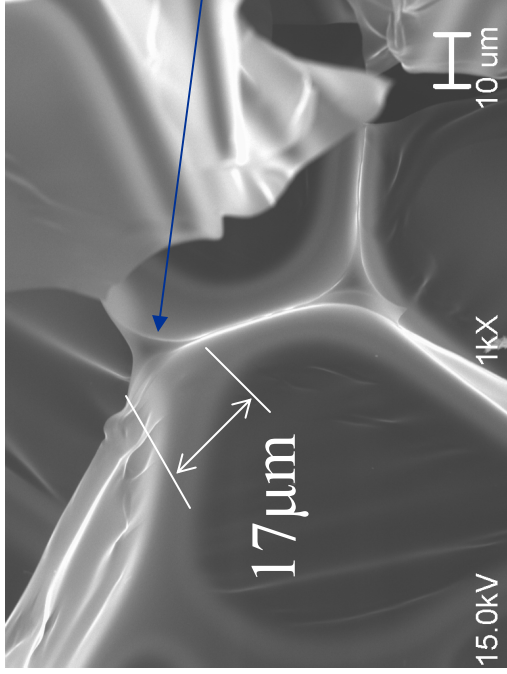
Packs to fill Volume



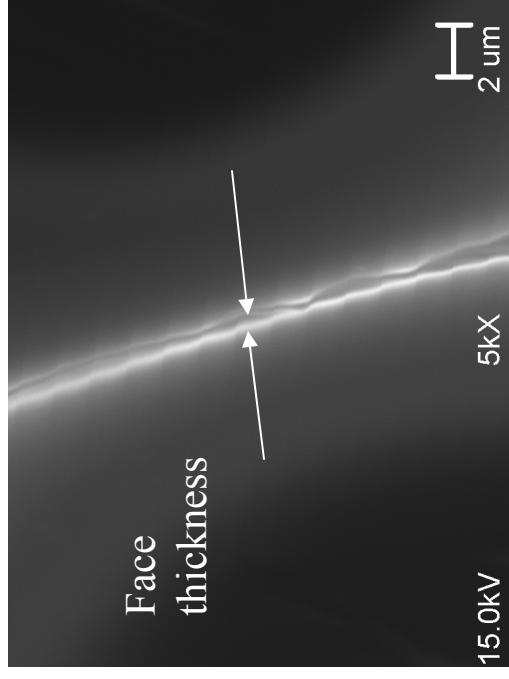


BX-265 Cell Dimensions

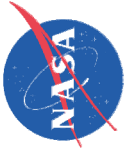
- Edges are solid, triangular beams.



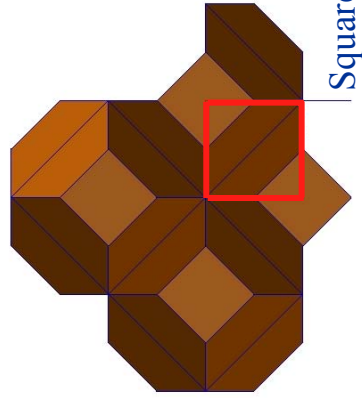
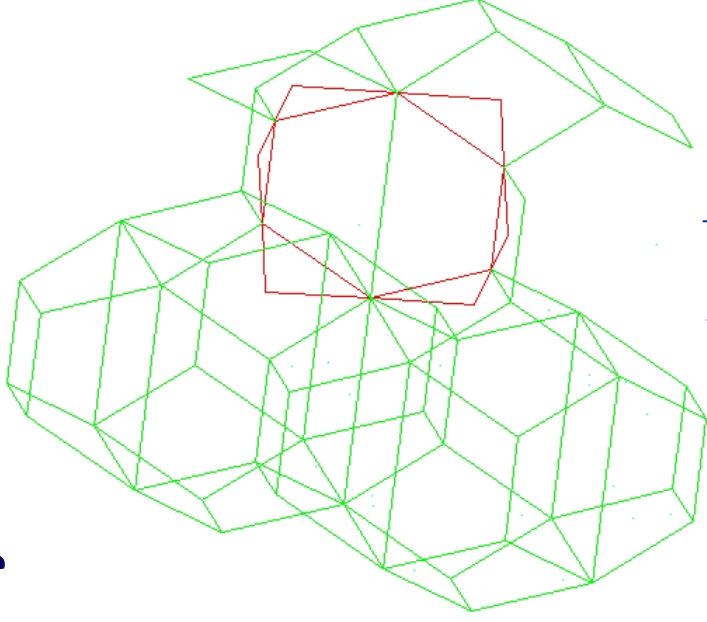
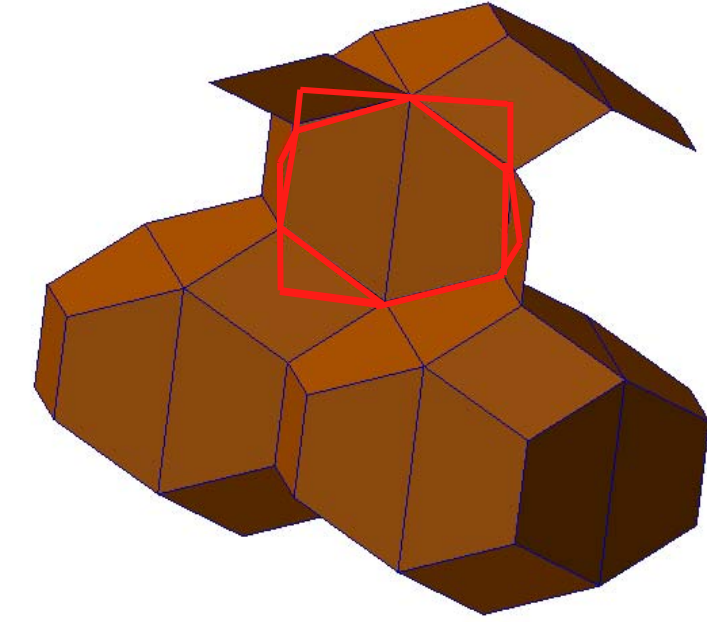
**Triangular edges are 17 µm in width
with average radii of 18 µm**



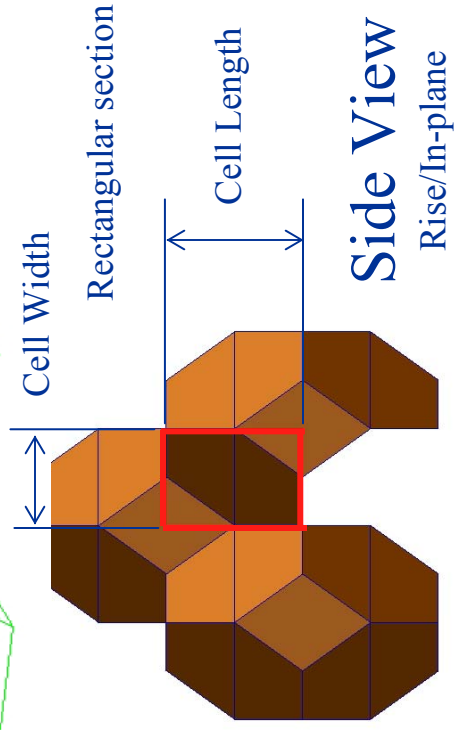
Faces are thin membranes 0.1 < t < 1.0 µm



Symmetry



Top View
Normal to Rise



Side View
Rise/In-plane

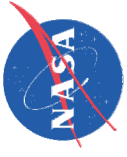
Cell Aspect Ratio
(Cell Length/Cell Width)

Cell Width

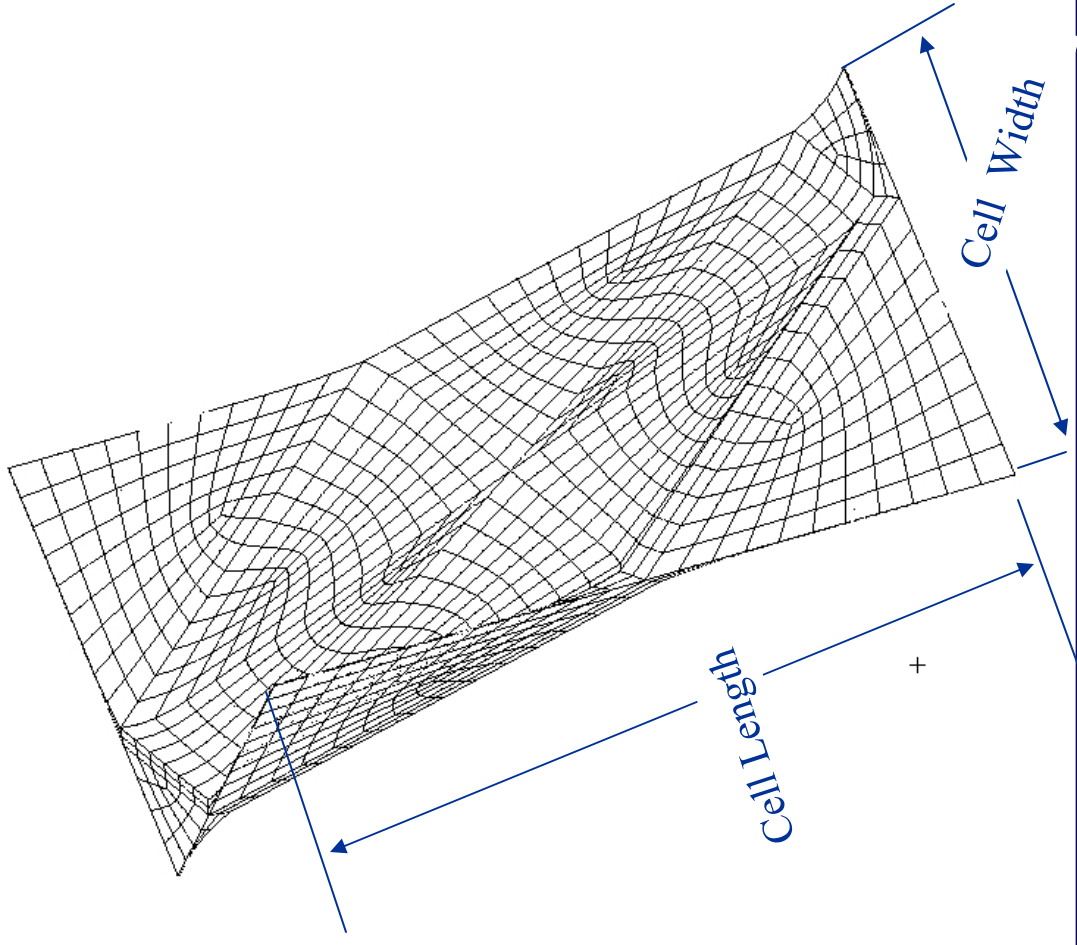
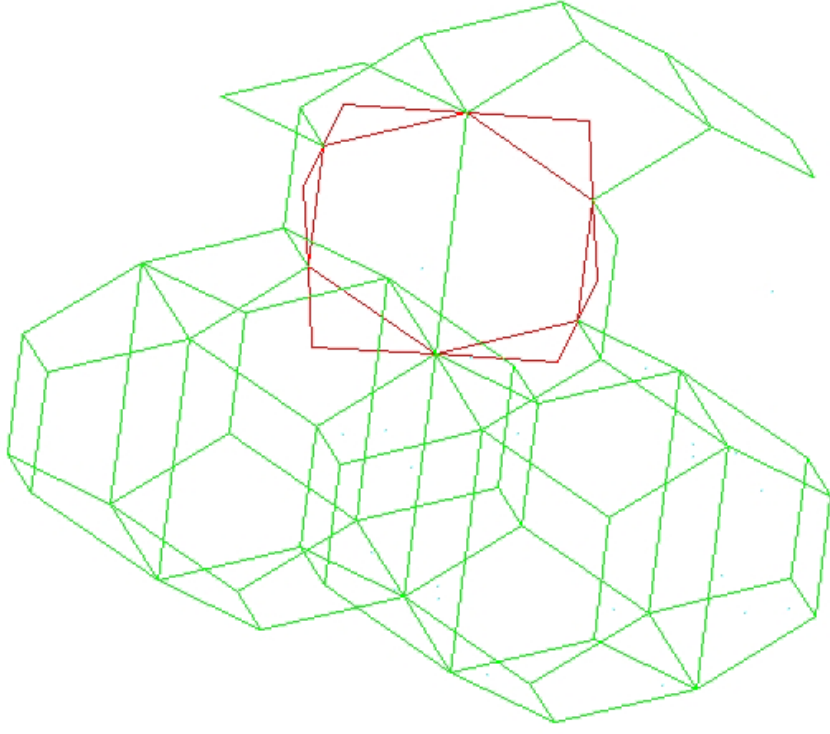
Rectangular section

Cell Length

Square section



Finite Element Mesh for 1/8th Symmetry



Rise Direction

Fix in X

Fix in Y

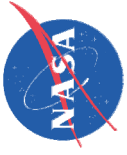
Half Wall Thickness

Full Wall Thickness

Half Wall Thickness

Fix in Z Along the bottom

15195 Nodes
2948 21-Node Brick Elements
wall thickness 0.5 μm
17 μm triangular edges
with 18 μm radii of curvatures



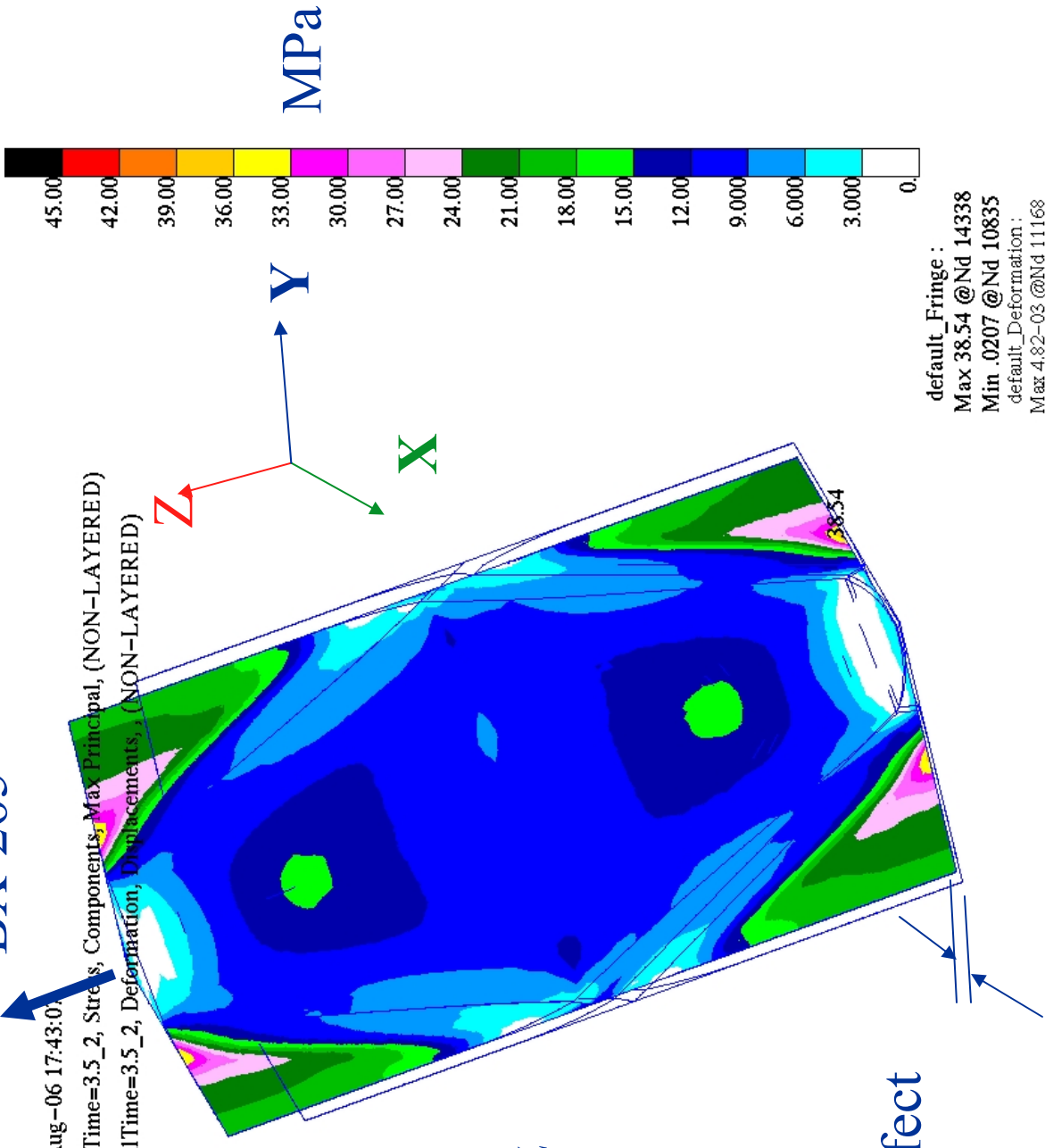
Maximum Principal Stress for loading in the rise direction

BX-265

MSC.Patran 2005 r2 08-Aug-06 17:43:01

Fringe: Static, Step2, TotalTime=3.5_2, Stress, Components, Max Principal, (NON-LAYERED)

Deform: Static, Step2, TotalTime=3.5_2, Deformation, Displacements,, (NON-LAYERED)



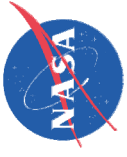
Loading direction

In

Rise Direction

0.31 MPa applied Stress in Z

Poisson's Effect

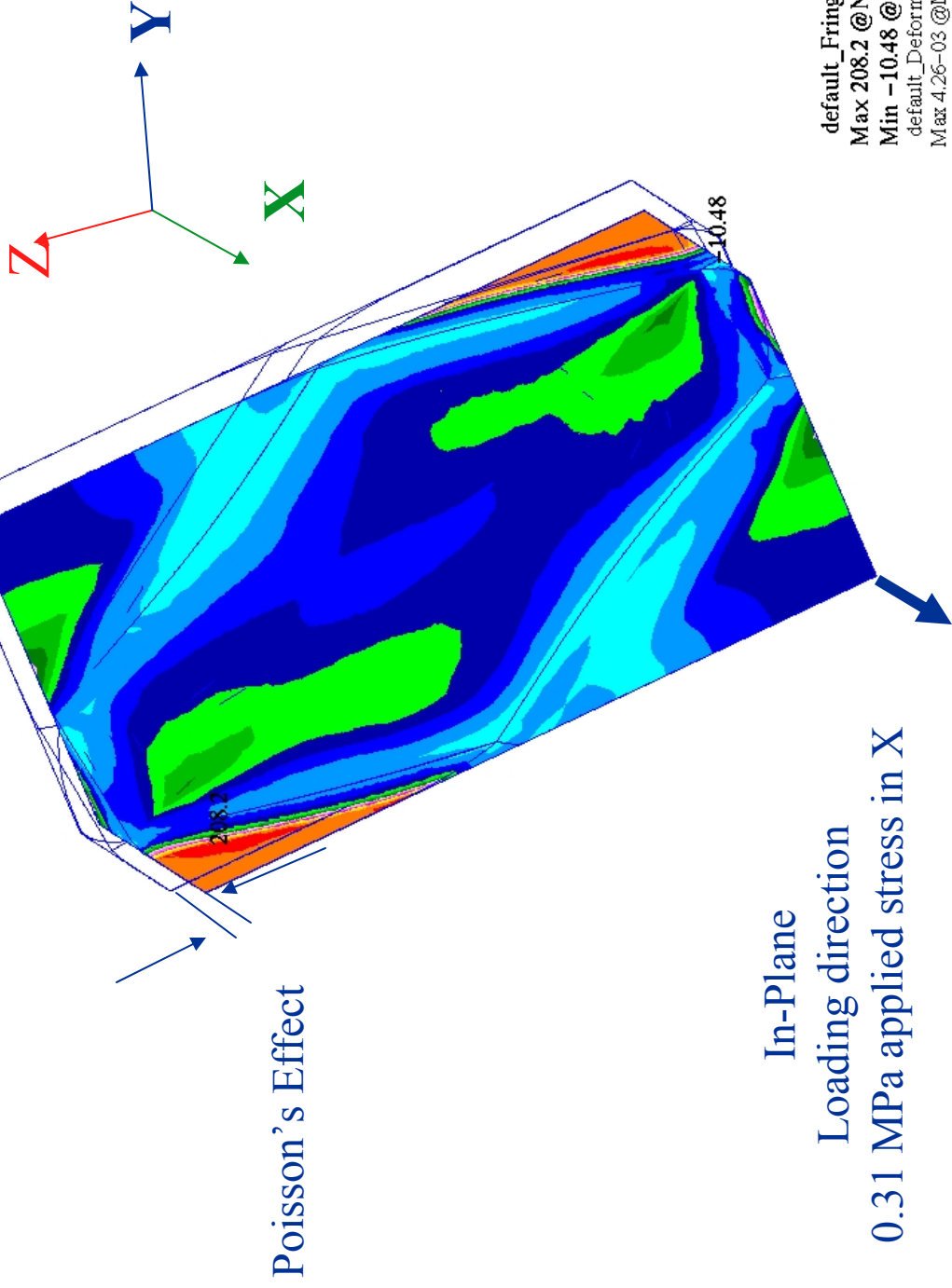


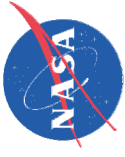
Maximum Principal Stress for in-plane loading BX-265

MSC.Patran 2005 r2 08-Aug-06 17:35:35

Fringe: Static, Step2, TotalTime=3.5, Stress, Components, Max Principal, (NON-LAYERED)

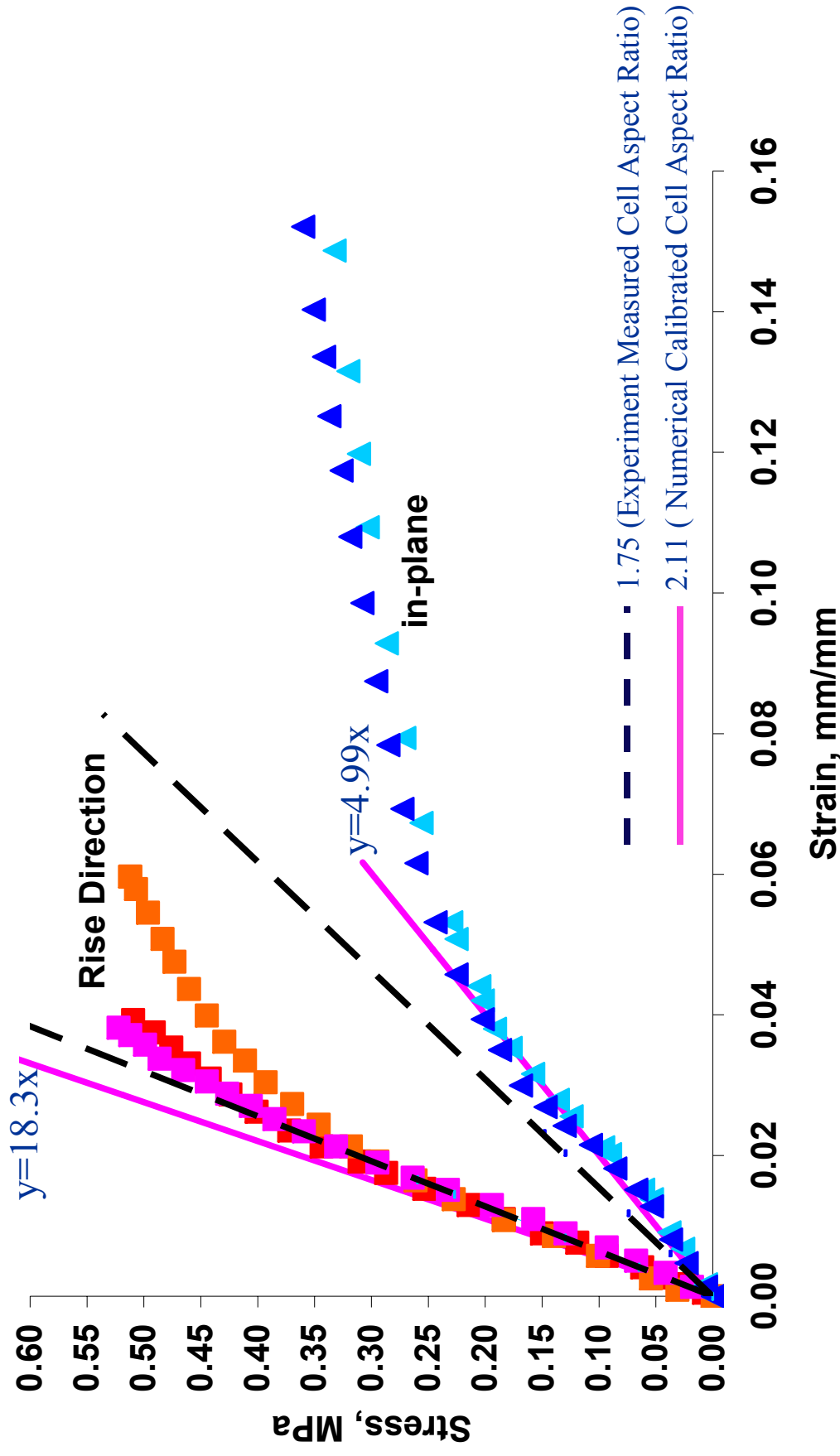
Deform: Static, Step2, TotalTime=3.5, Deformation, Displacements,



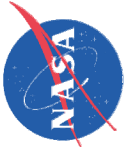


Calibration of BX-265 Stress/Strain Curve

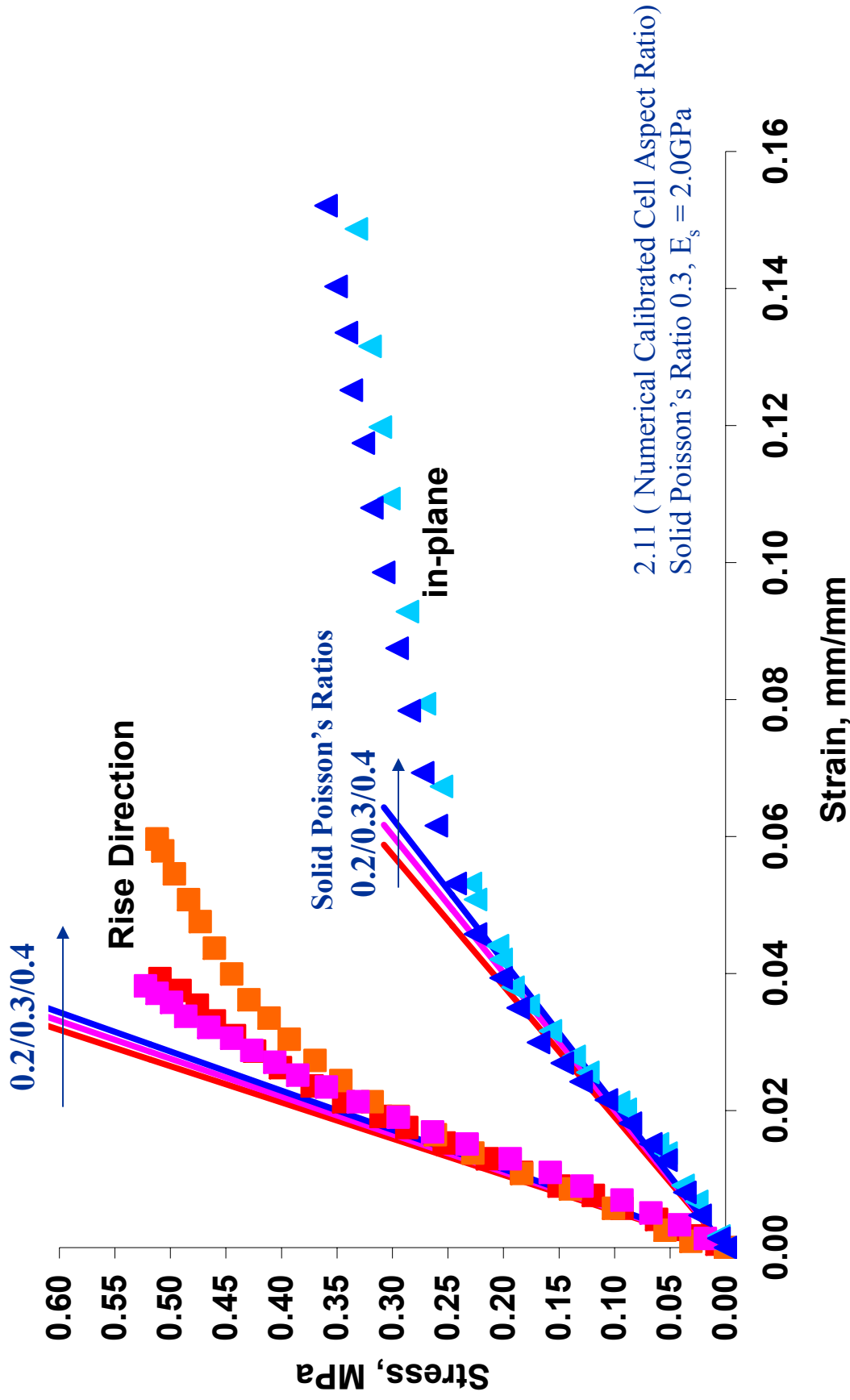
BX-265 @ room temperature



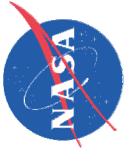
Experimental data courtesy of LMSSC External Tank Return to Flight, Fracture Toughness: Phase 2 (Kristin Morgan)



Effects of Solid Poisson's Ratio on the Foam Stress/Strain Curve BX-265 @ room temperature

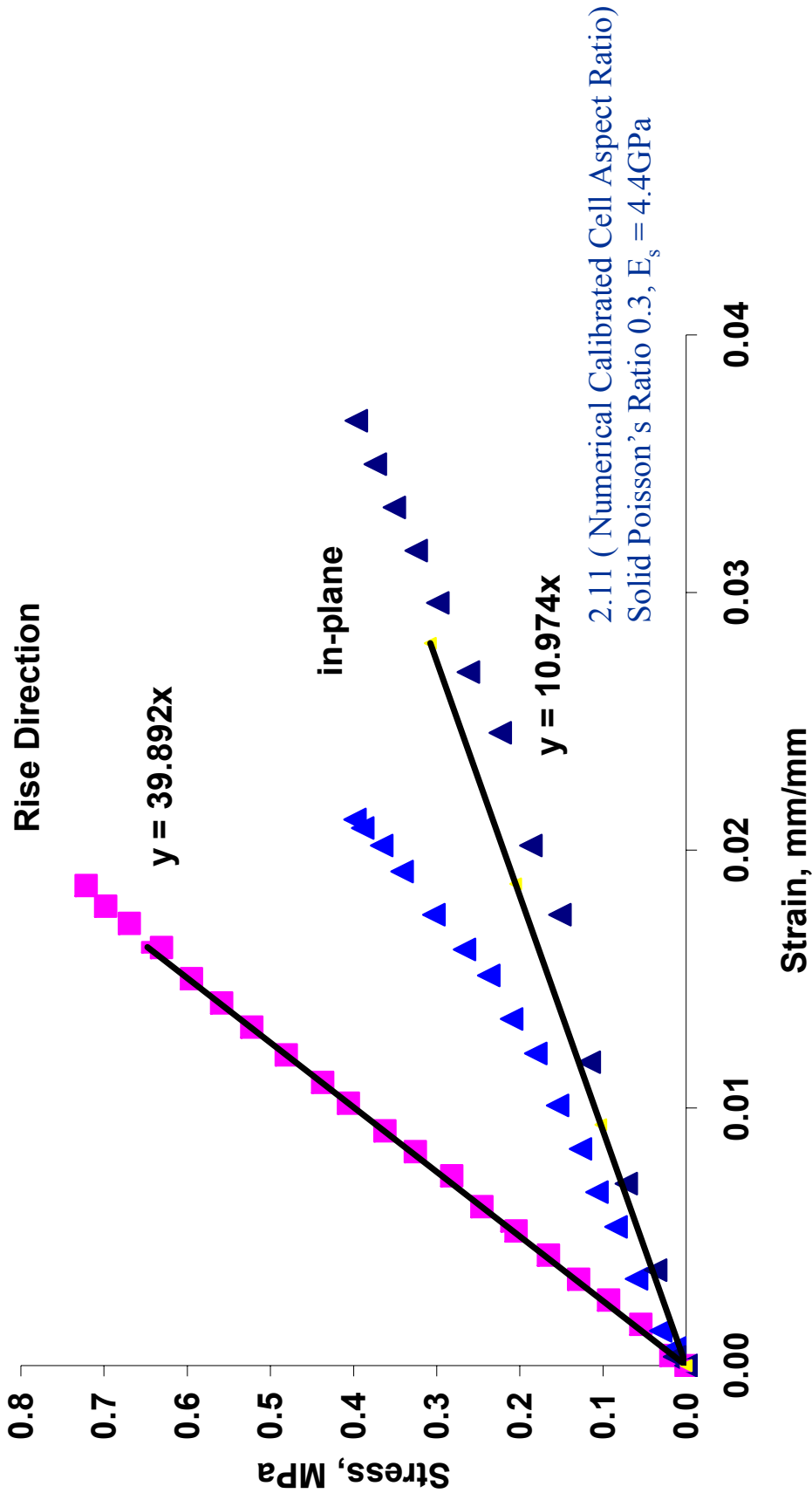


Experimental data courtesy of LMSSC External Tank Return to Flight, Fracture Toughness: Phase 2 (Kristin Morgan)

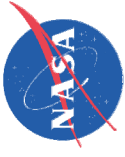


Calibration of BX-265 Stress/Strain Curve (-320 F)

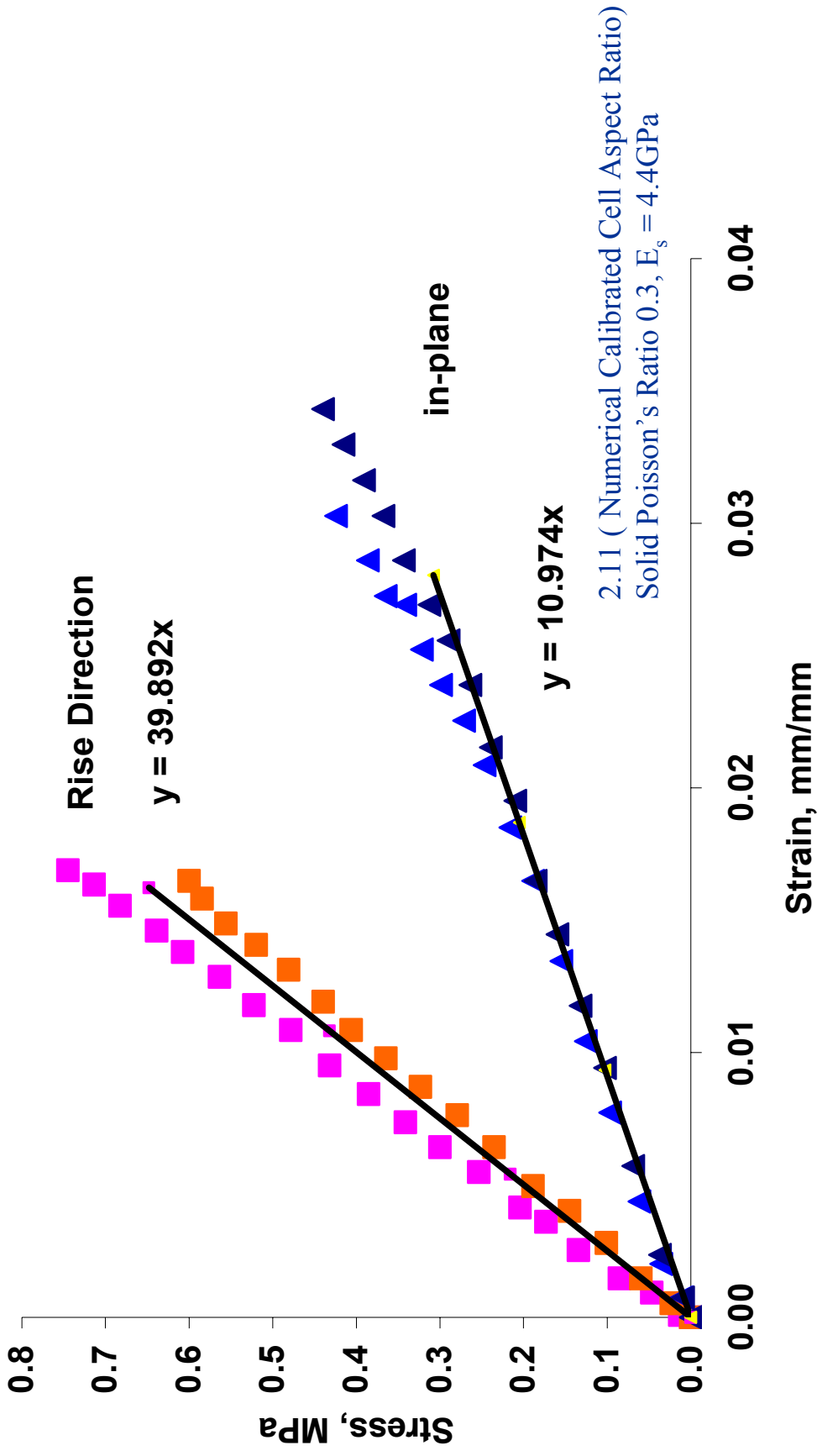
BX-265 @ -195.6 C



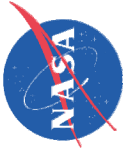
Experimental data courtesy of LMSSC External Tank Return to Flight, Fracture Toughness: Phase 2 (Kristin Morgan)



Calibration of BX-265 Stress/Strain Curve (-420 F) BX-265 @ -251 C

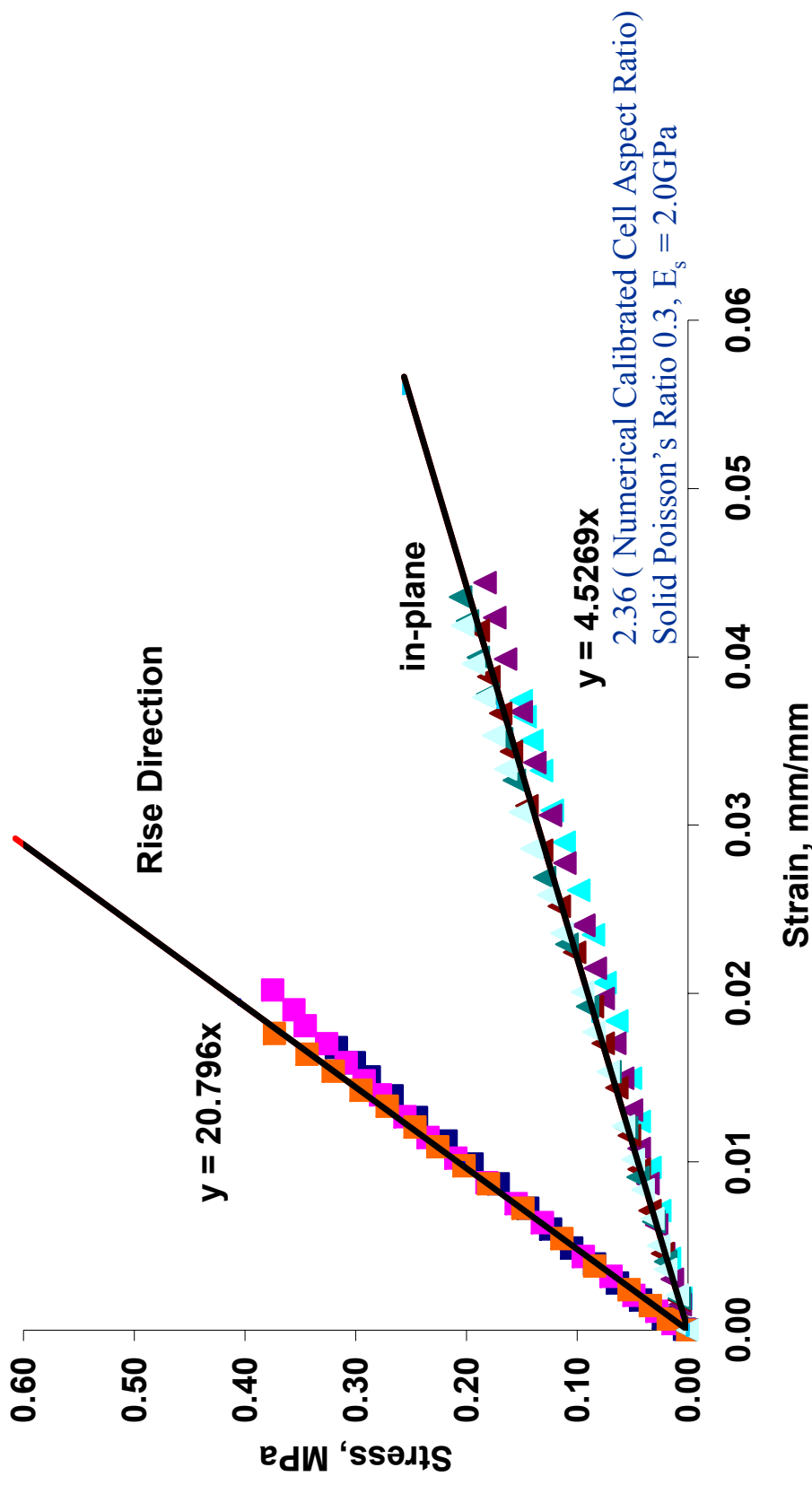


Experimental data courtesy of LMSSC External Tank Return to Flight, Fracture Toughness: Phase 2 (Kristin Morgan)

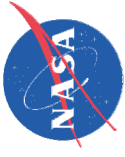


Calibration of NCFI Stress/Strain Curve (RT)

NCFI-24-124 @ room temperature

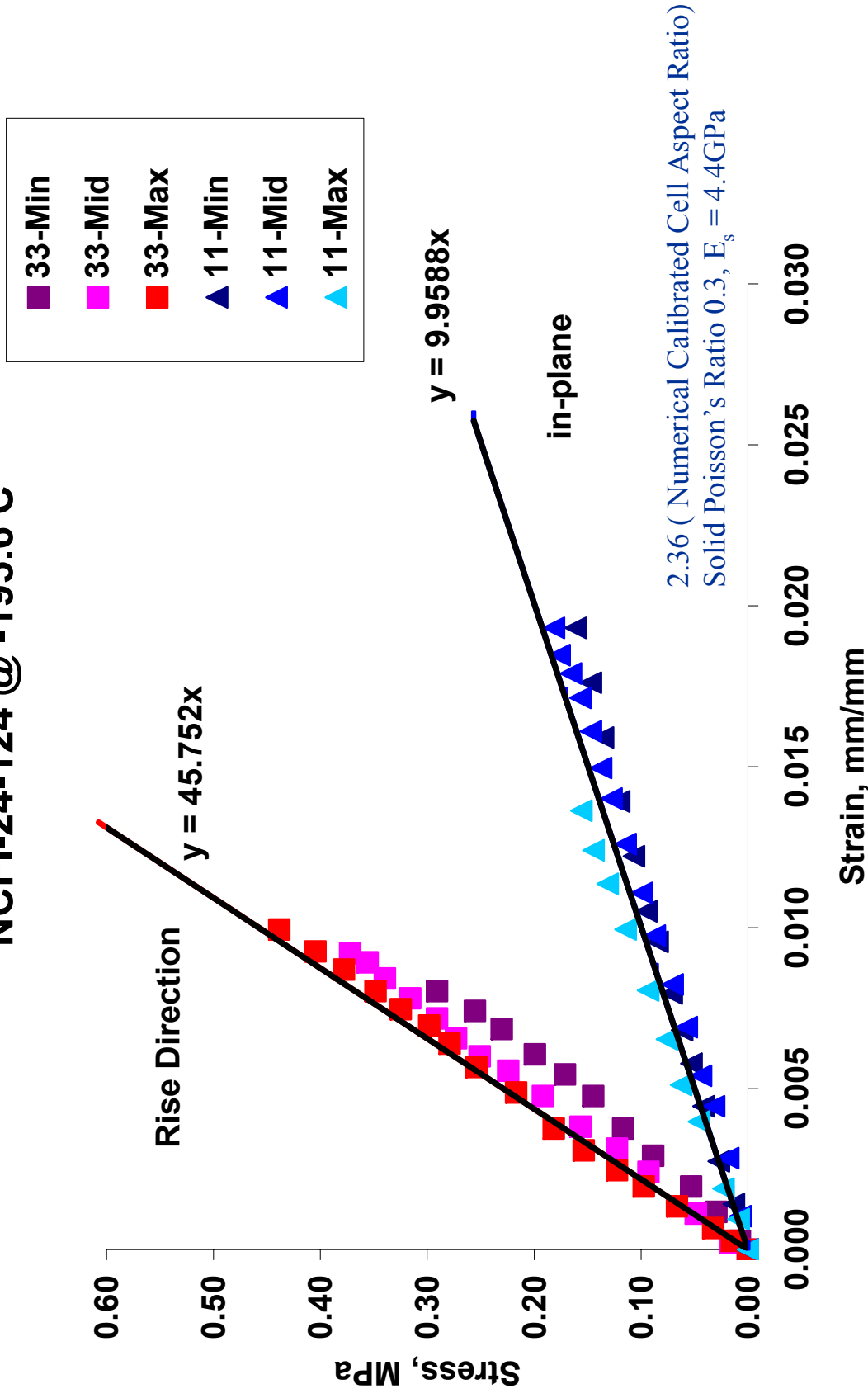


Experimental data courtesy of LMSSC External Tank Return to Flight, Fracture Toughness: Phase 2 (Kristin Morgan)

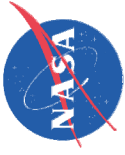


Calibration of NCFI Stress/Strain Curve (-320 F)

NCFI-24-124 @ -195.6 C

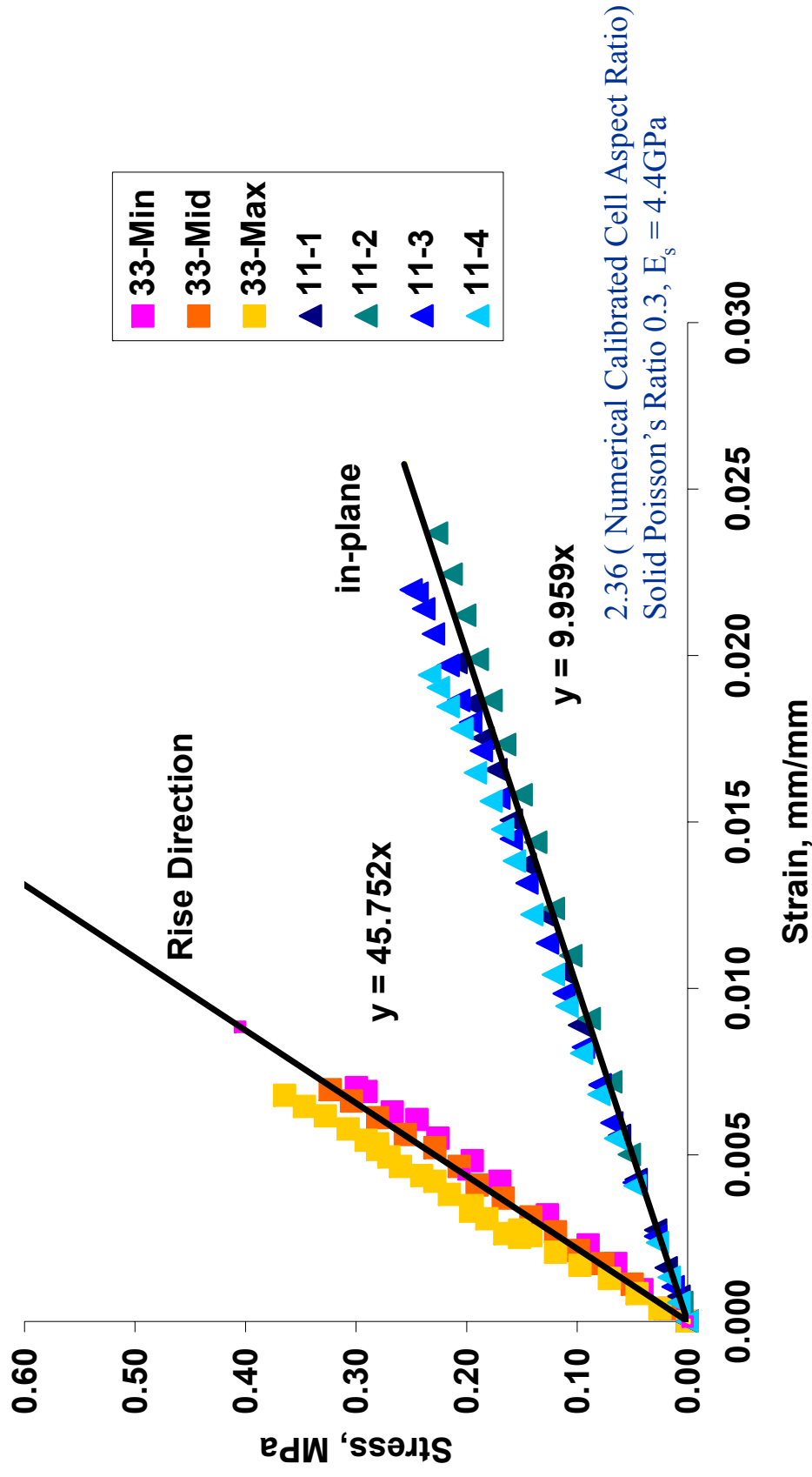


Experimental data courtesy of LMSSC External Tank Return to Flight, Fracture Toughness: Phase 2 (Kristin Morgan)

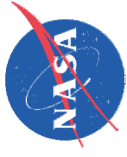


Calibration of NCFI Stress/Strain Curve (-420 F)

NCFI-24-124 @ -251 C

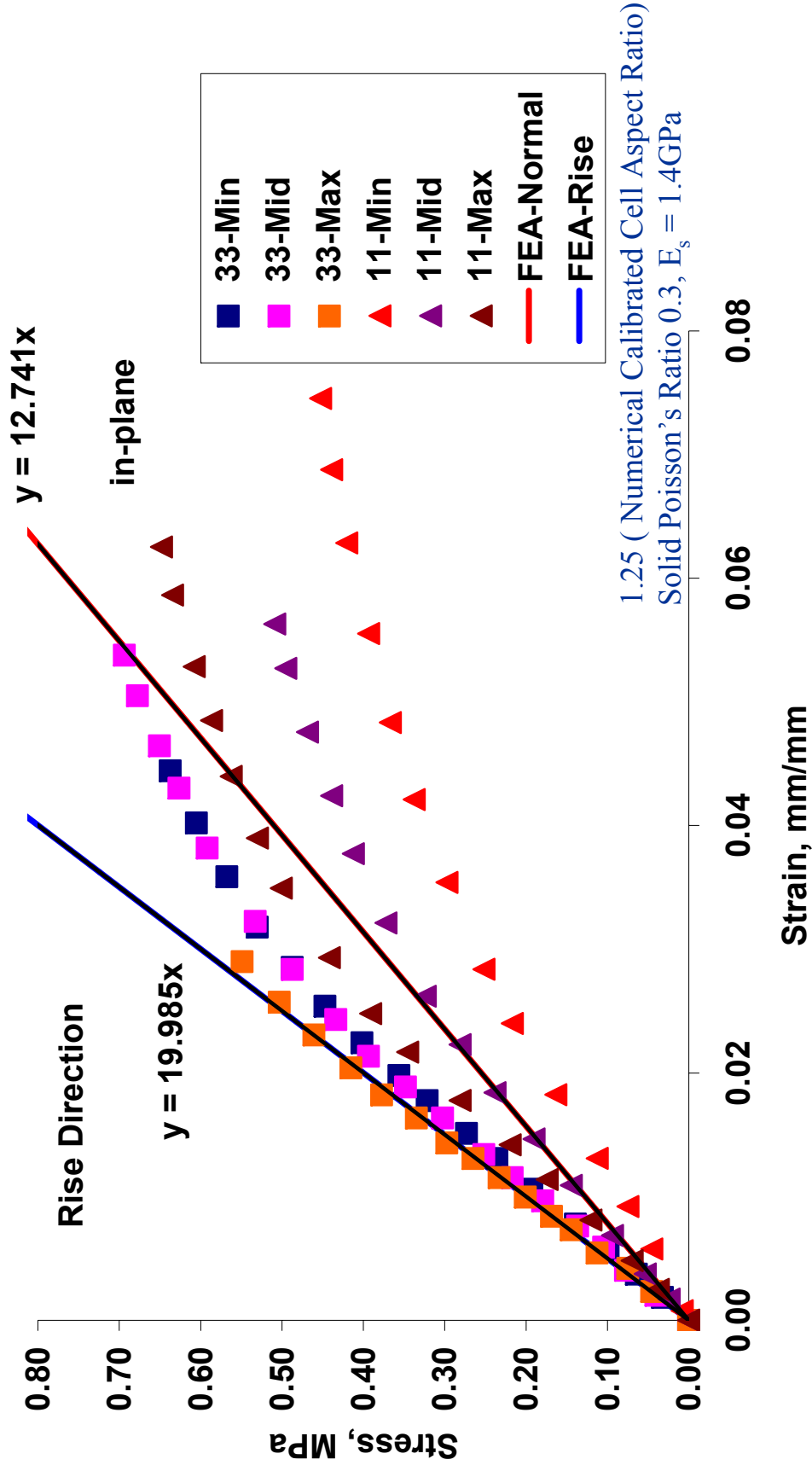


Experimental data courtesy of LMSSC External Tank Return to Flight, Fracture Toughness: Phase 2 (Kristin Morgan)

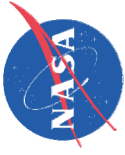


Calibration of PDL Stress/Strain Curve (RT)

PDL-1034 @ room temperature

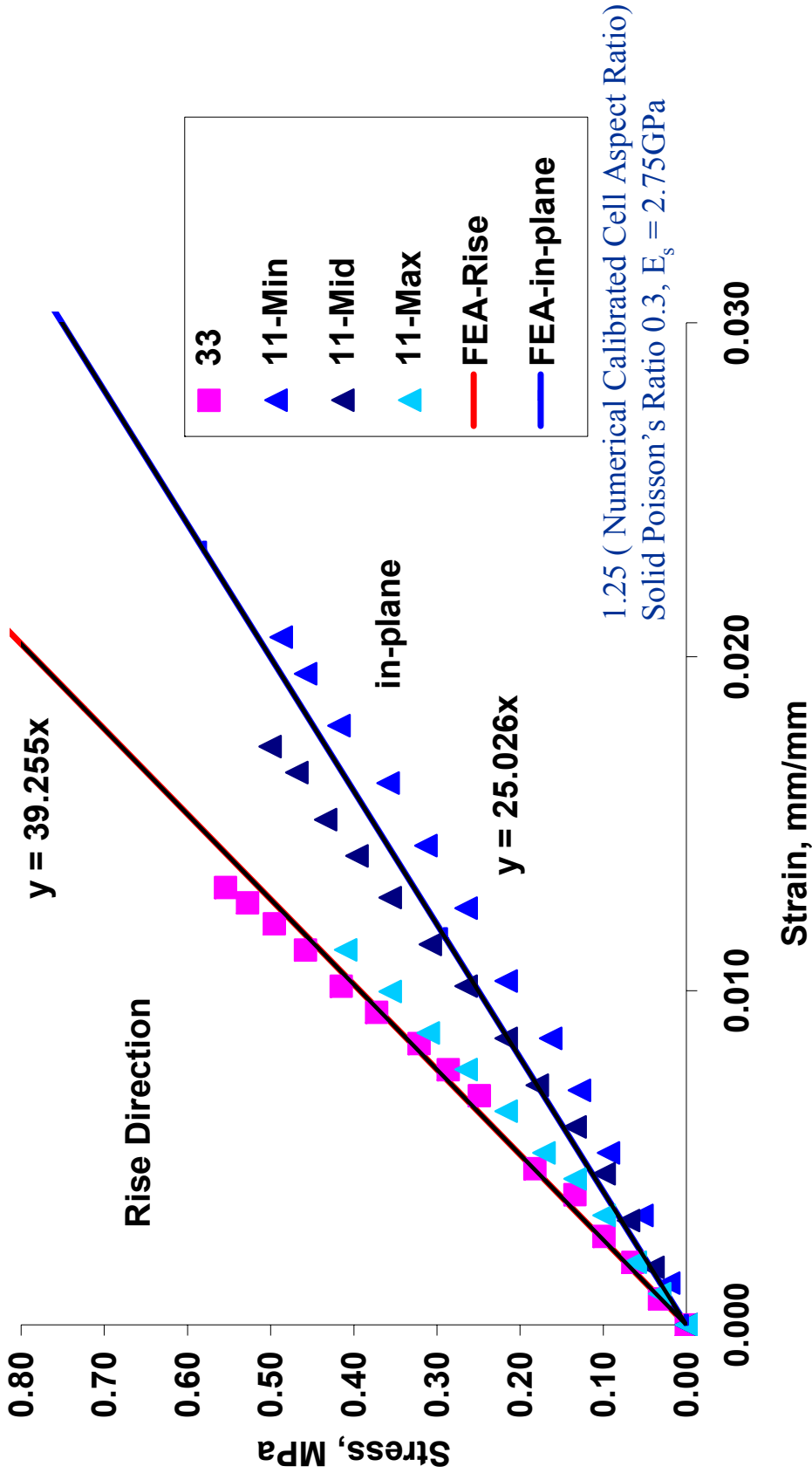


Experimental data courtesy of LMSSC External Tank Return to Flight, Fracture Toughness: Phase 2 (Kristin Morgan)

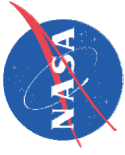


Calibration of PDL Stress/Strain Curve (-320 F)

PDL-1034 @ -195.6 C

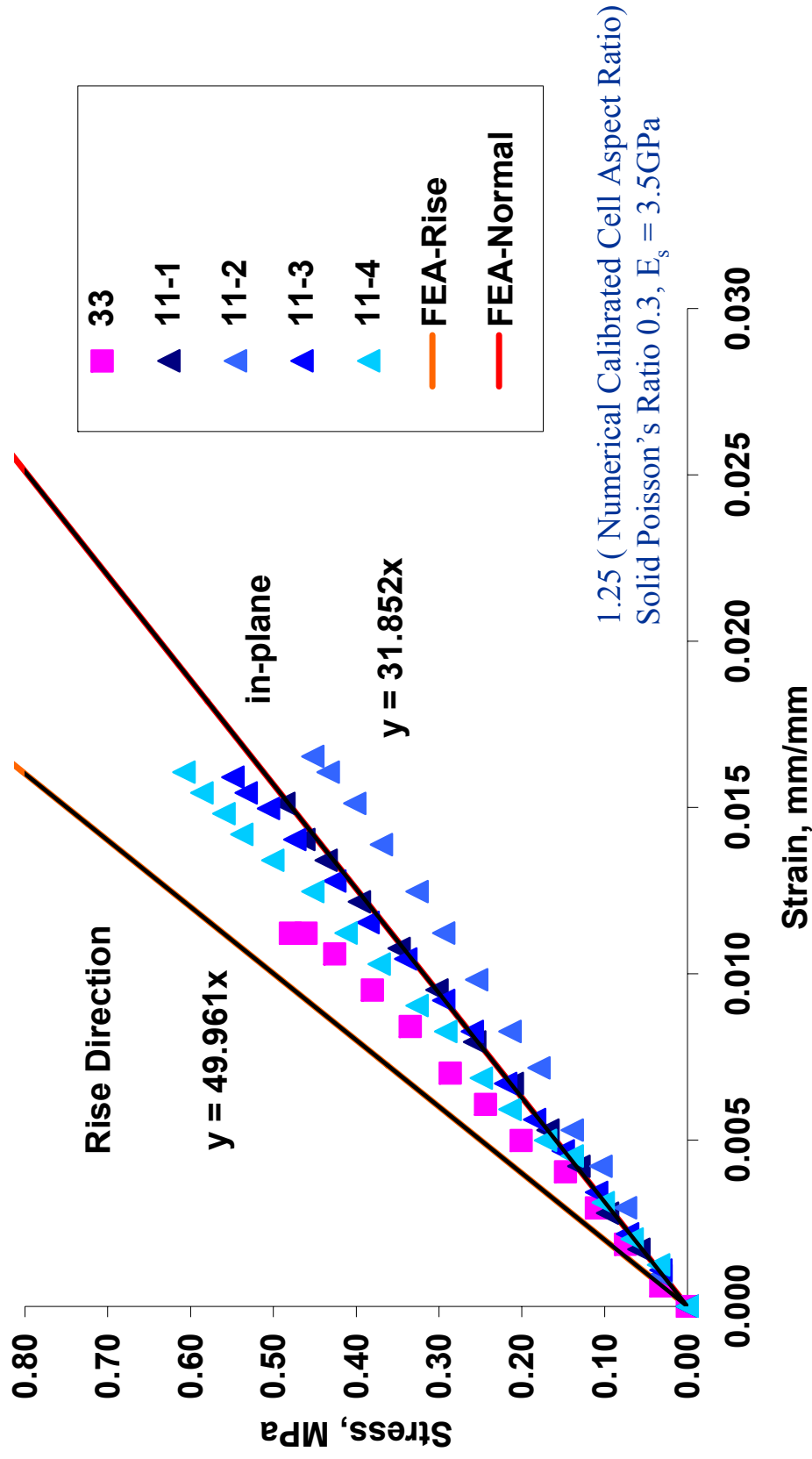


Experimental data courtesy of LMSSC External Tank Return to Flight, Fracture Toughness: Phase 2 (Kristin Morgan)

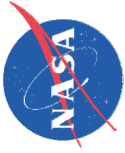


Calibration of PDL Stress/Strain Curve (-420 F)

PDL-1034 @ -251 C



Experimental data courtesy of LMSSC External Tank Return to Flight, Fracture Toughness: Phase 2 (Kristin Morgan)



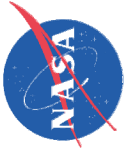
	Average Unit Cell			Solid			
	Length	Width	Aspect Ratio	Poisson's Ratio	Modulus, E_s		
	μm	μm			@ RT	@ -195.6 C	@ -195.6 C
					GPa	GPa	GPa
Calibrated	BX-265	262	124	2.11	0.3	4.4	4.4
	NCFI	236	100	2.36	0.3	4.4	4.4
	PDL	294	235	1.25	0.3	2.75	3.5
Measured	BX-265	218	124	1.76			

Foam Effective Elastic Properties

BX-265							
Temp (C)	E11 (MPa)	E22 (MPa)	E33 (MPa)	nu12	nu13	nu23	
-251	10.97	10.97	39.89	0.488	0.143	0.143	
-196	10.97	10.97	39.89	0.488	0.143	0.143	
24	4.99	4.99	18.13	0.488	0.143	0.143	

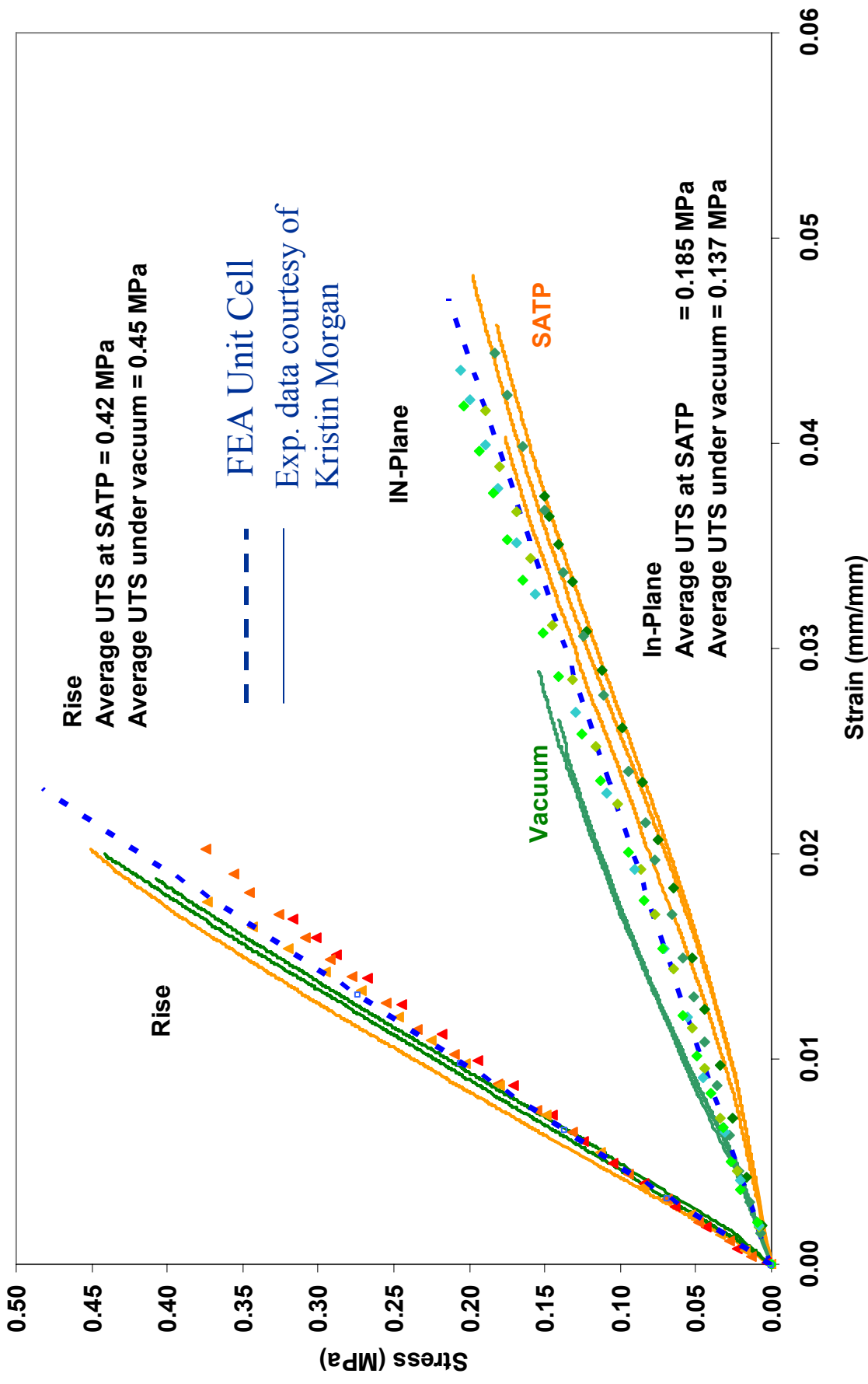
NCFI							
Temp (C)	E11 (MPa)	E22 (MPa)	E33 (MPa)	nu12	nu13	nu23	
-251	9.96	9.96	45.75	0.528	0.114	0.114	
-196	9.96	9.96	45.75	0.528	0.114	0.114	
24	4.53	4.53	20.80	0.528	0.114	0.114	

PDL							
Temp (C)	E11 (MPa)	E22 (MPa)	E33 (MPa)	nu12	nu13	nu23	
-253	31.85	31.85	49.96	0.362	0.249	0.249	
-196	25.03	25.03	39.26	0.362	0.249	0.249	
24	12.74	12.74	19.99	0.362	0.249	0.249	

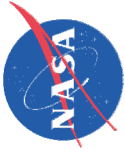


UTS of NCFI at RT

NCFI 24-124: Round 7" Dogbone Specimen



Symbols data LMSSC External Tank Return to Flight, Fracture Toughness: Phase 2 (Kristin Morgan)



Maximum Principal Stress for loading in the rise direction

NCFI (1 atm)

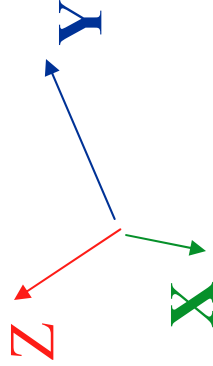
MSC.Patran 2005 r2 29-Jun-06 10:34:06

Fringe: Static, Step2, TotalTime=2., Stress, Components: Max Principal, (NON-LAYERED)

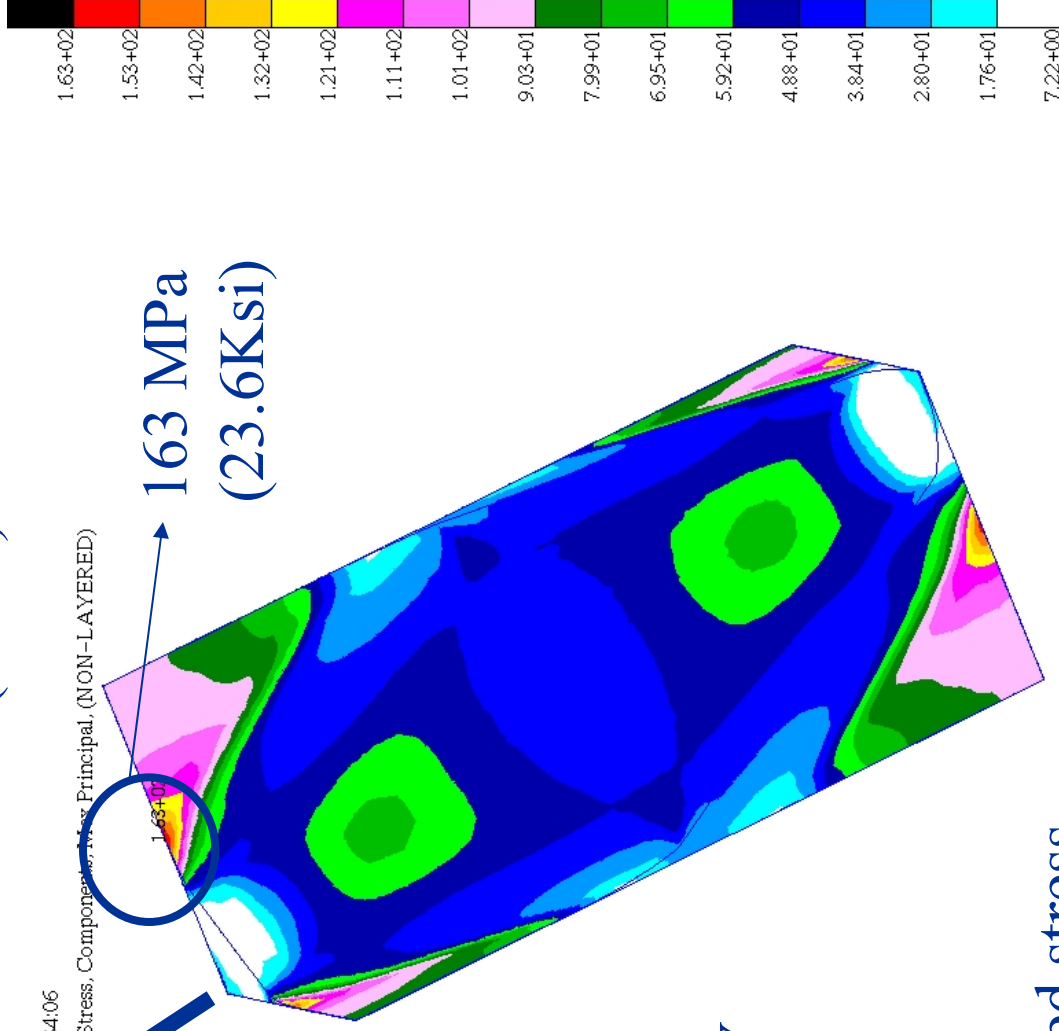
MPa

163 MPa
(23.6Ksi)

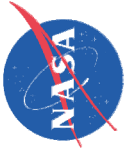
Unit Cell
Loading
In Rise Direction



Rise at 1-atm
@ 0.42 MPa applied stress



default_Fringe :
Max 1.63+02 @Nd 11200
Min 7.22+00 @Nd 976

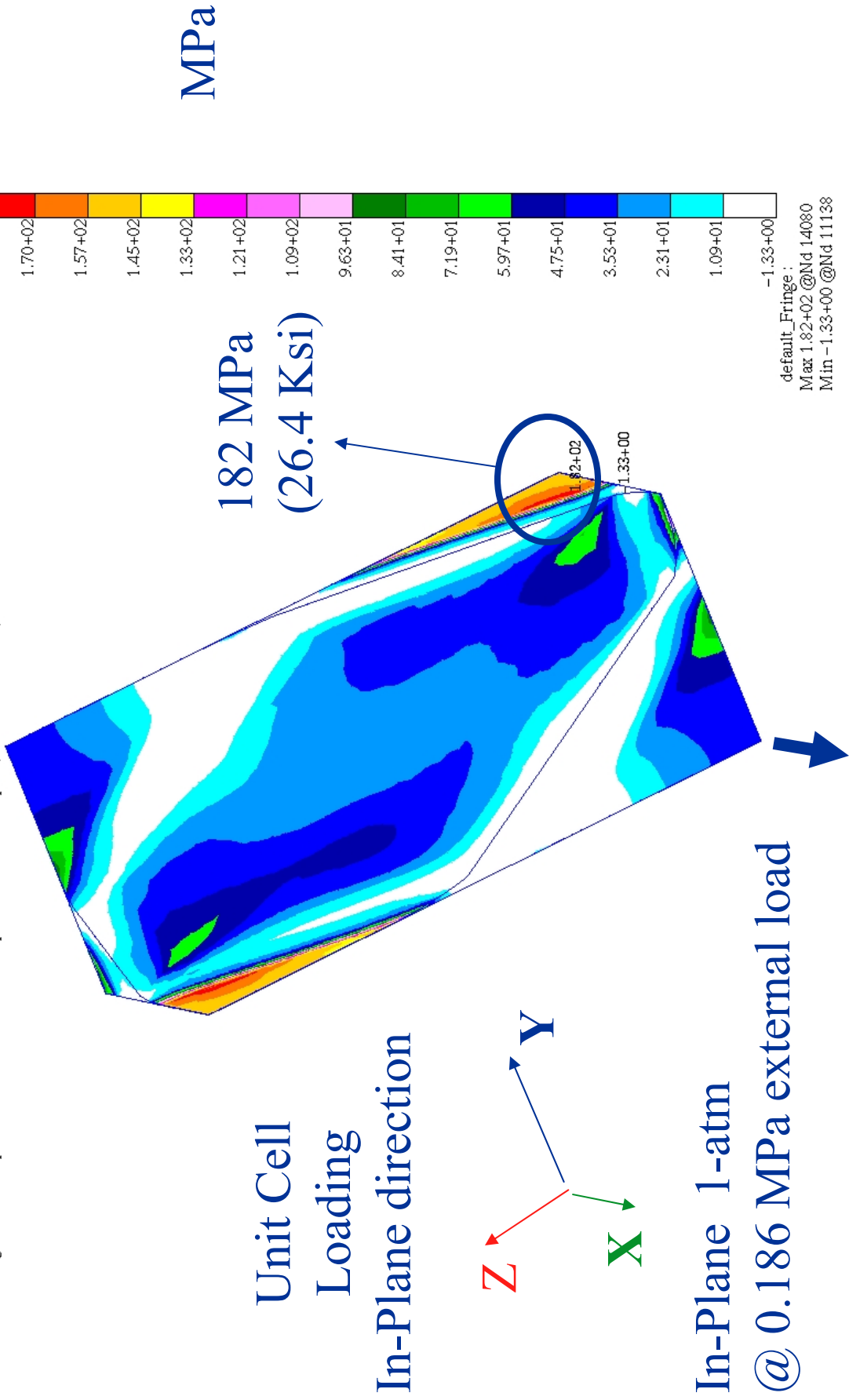


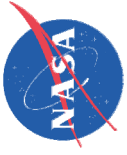
Maximum Principal Stress for loading in the in-plane direction

NCFI (1 atm)

MSC.Patran 2005 r2 29-Jun-06 12:32:09

Fringe: Static, Step2, TotalTime=2., Stress, Components, Max Principal, (NON-LAYERED)



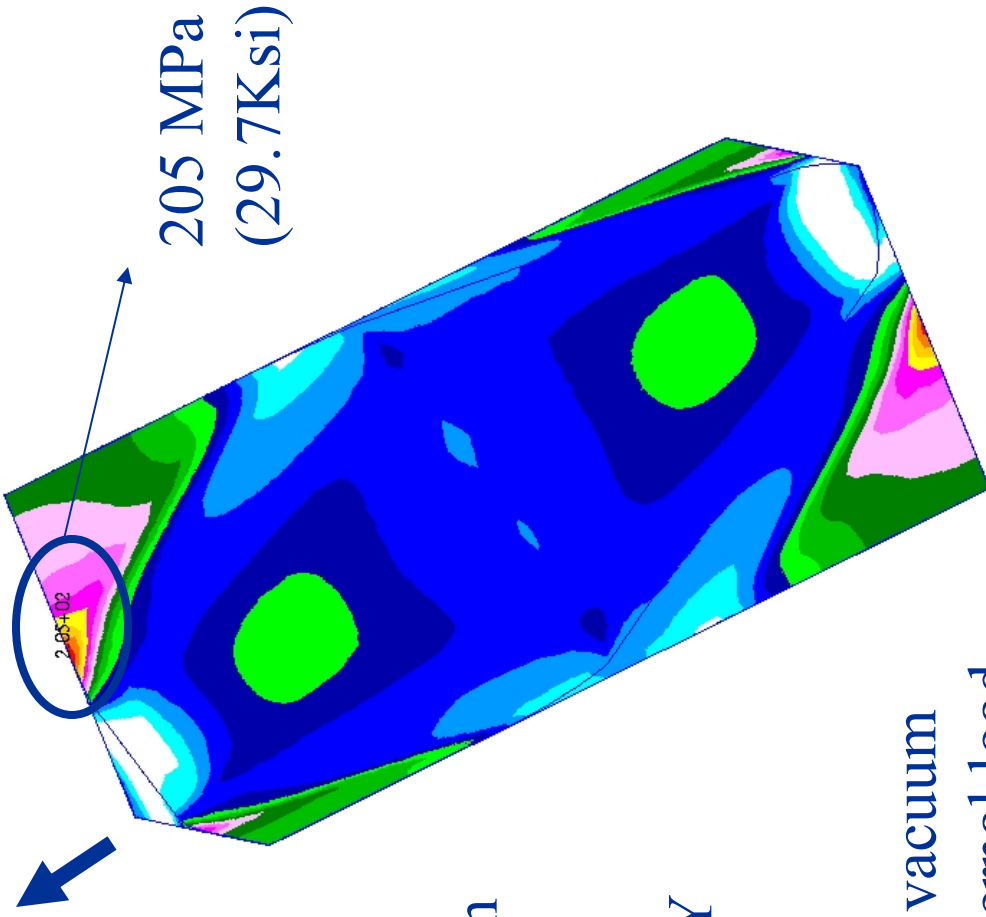


Maximum Principal Stress for loading in the rise direction

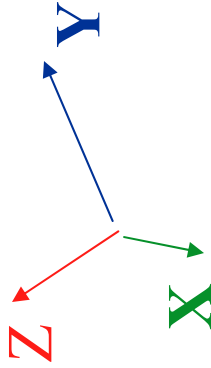
NCFI (Vacuum Test)

MSC.Patran 2005 r2 29-Jun-06 12:22:42

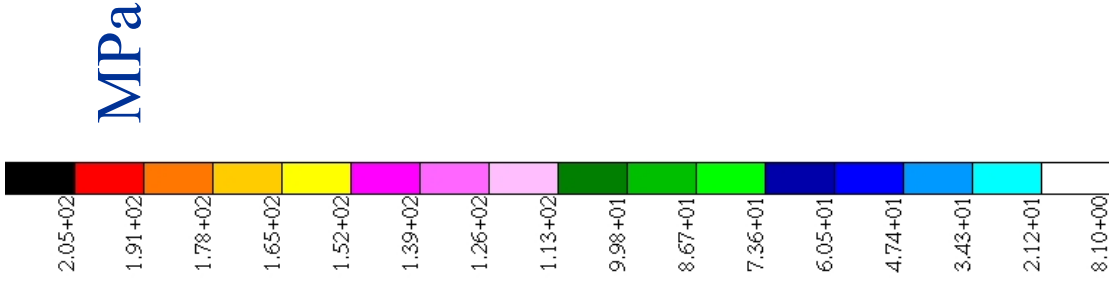
Fringe: Static, Step2,TotalTime=2., Stress, Components, Max Principal, (NON-LAYERED)



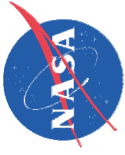
Unit Cell
Loading
In Rise Direction



Rise mid-cell in vacuum
@ 0.45 MPa external load



default_Fringe:
Max 2.05+02 @Nd 11200
Min 8.10+00 @Nd 976



Maximum Principal Stress for loading in the rise direction NCFI (Vacuum Test)

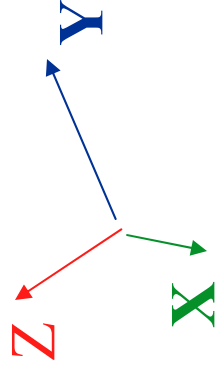
MSC.Patran 2005 r2 29-Jun-06 12:26:36

Fringe: Static, Step2,TotalTime=2., Stress, Components, Max Principal, (NON-LAYERED)

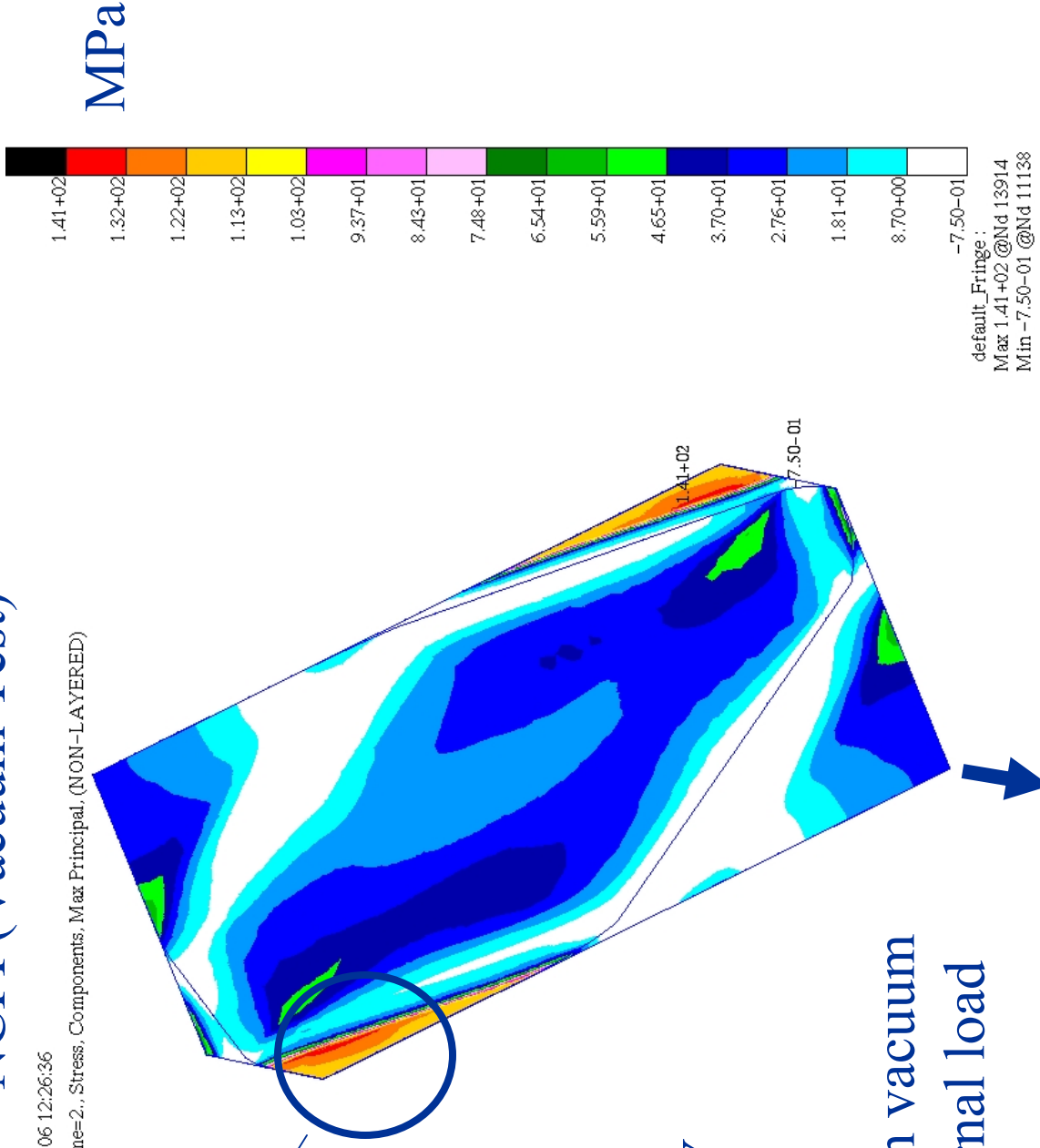
141 MPa
(20.5 Ksi)

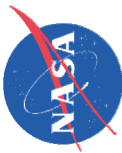
Unit Cell
Loading

In-Plane direction
vacuum



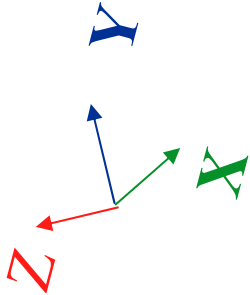
In-Plane mid-cell in vacuum
@ 0.127 MPa external load



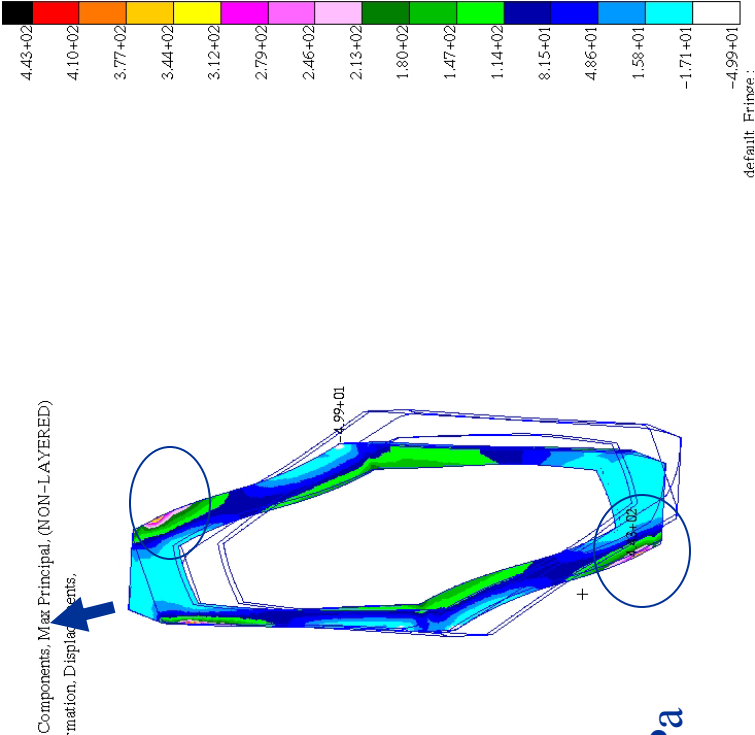


NCFI, UTS		Applied Stress	Max. Principal
		MPa	MPa
Rise	SATM	0.42	169
In-Plane	SATM	0.186	182
Rise	Vacuum	0.45	205
In-Plane	Vacuum	0.127	141

Loading in Rise Direction
Without cell walls (Open Cell)

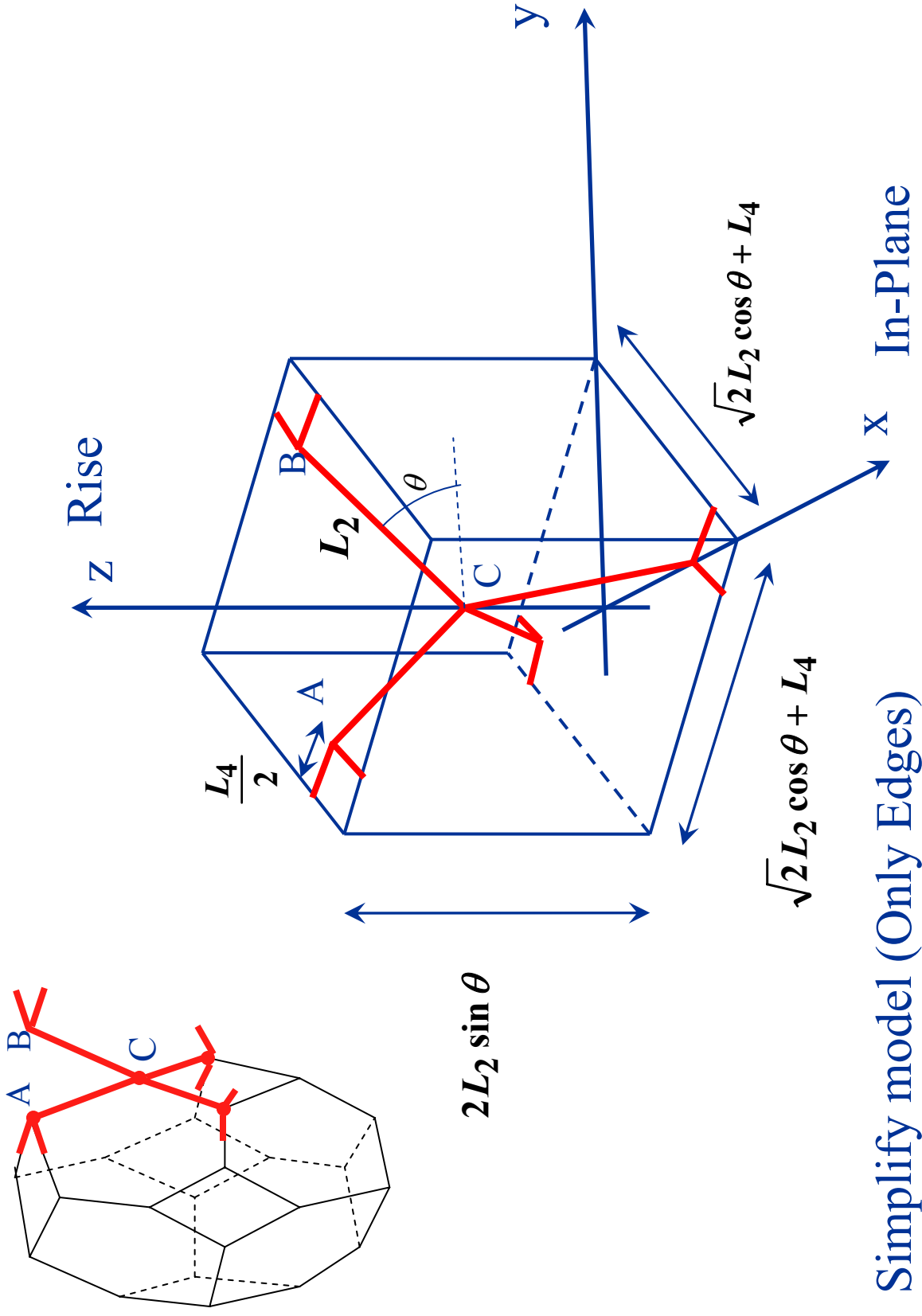


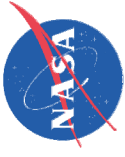
Max. Principal, 443MPa





Open Cell Structure For Failure Analysis

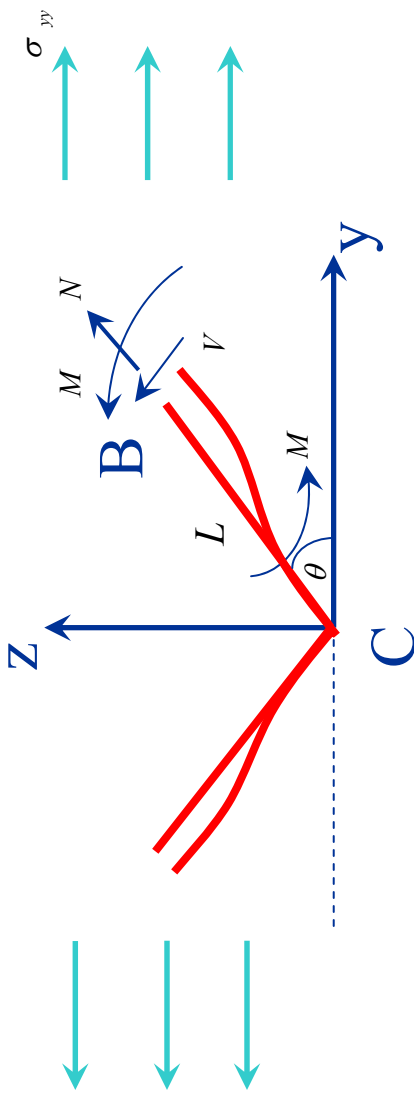
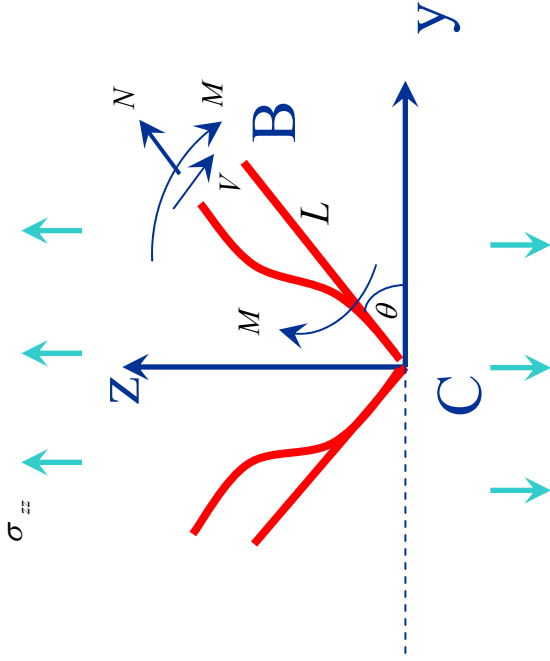




Equilibrium and Structural Analysis of Frames yields

$$N = \frac{\sigma_{zz} \sin \theta \left[\sqrt{2} L_2 \cos \theta + L_4 \right]^2}{2}$$

$$M = \frac{\sigma_{zz} L_2 \cos \theta \left[\sqrt{2} L_2 \cos \theta + L_4 \right]^2}{4}$$



$$N = \sigma_{yy} L_2 \cos \theta \sin \theta \left[2 L_2 \cos \theta + \sqrt{2} L_4 \right]$$

$$M = \frac{\sigma_{yy} L_2^2 \sin^2 \theta \left[2 L_2 \cos \theta + \sqrt{2} L_4 \right]}{2}$$



Strength of Materials approach for calculating the maximum stress at the outer fiber of an axial and flexural of the edges

$$\sigma_{\max} = \frac{N}{A} + \frac{Mr}{I}$$

Where

A = edge x-area

I = edge moment of Inertia

r = distance to outer fiber

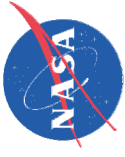
Leads to the relation between the maximum stress in the struts and the average applied stress in the z and y directions:

$$\sigma_{zz} = \frac{\sigma_{\max}}{\left[\frac{\sin \theta}{2A} + \frac{L_2 r \cos \theta}{4I} \right] \left(\sqrt{2} L_2 \cos \theta + L_4 \right)^2}$$

$$\sigma_{yy} = \frac{\sigma_{\max}}{\left[\frac{L_2 \cos \theta \sin \theta}{A} + \frac{L_2^2 r \sin^2 \theta}{2I} \right] \left(2L_2 \cos \theta + \sqrt{2} L_4 \right)}$$

Calculating the average cell strain and stress can also lead to determining the effective cell modulus in two directions :

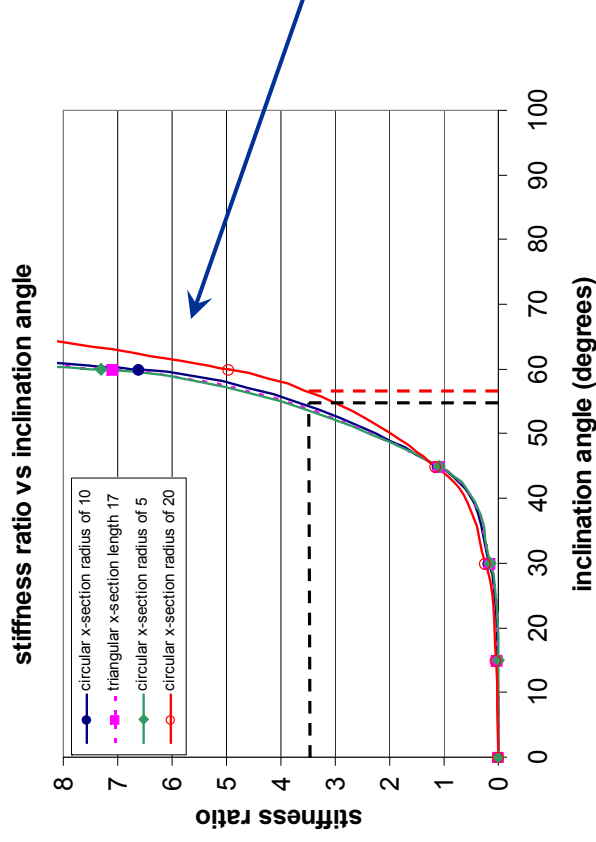
$$E_z = \frac{24}{L_2^2} \frac{EI \sin \theta}{\cos^2 \theta + \frac{12I \sin^2 \theta}{AL_2^2}} \quad E_y = \frac{12}{\left[2L_2^3 \sin^2 \theta + L_4^3 + \frac{12I}{A} \left(2L_2 \cos^2 \theta + L_4 \right) \right]} \frac{EI}{L_2 \sin \theta}$$



Ratio of the Stiffness in the z- and y-directions is

$$\frac{E_z}{E_y} = \frac{2 \sin^2 \theta \left[2L_2^3 \sin^2 \theta + L_4^3 + \frac{12I}{A} (2L_2 \cos^2 \theta + L_4) \right]}{L_2 \left[\cos^2 \theta + \frac{12I \sin^2 \theta}{AL_2^2} \right] \left[\sqrt{2} L_2 \cos \theta + L_4 \right]^2}$$

Assuming L_2 and L_4 are 69 μm and 28 μm , respectively, we can plot the stiffness ratio versus inclination angle.

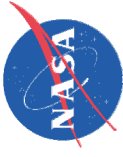


Measured stiffness ratio for BX-265 at room temperature

$$\frac{E_z}{E_y} = \frac{18 \text{ MPa}}{5 \text{ MPa}} = 3.6$$

Strut cross-section has small influence on stiffness ratio versus inclination angle

Results indicate that the inclination angle for BX-265 is $\sim 55^\circ$

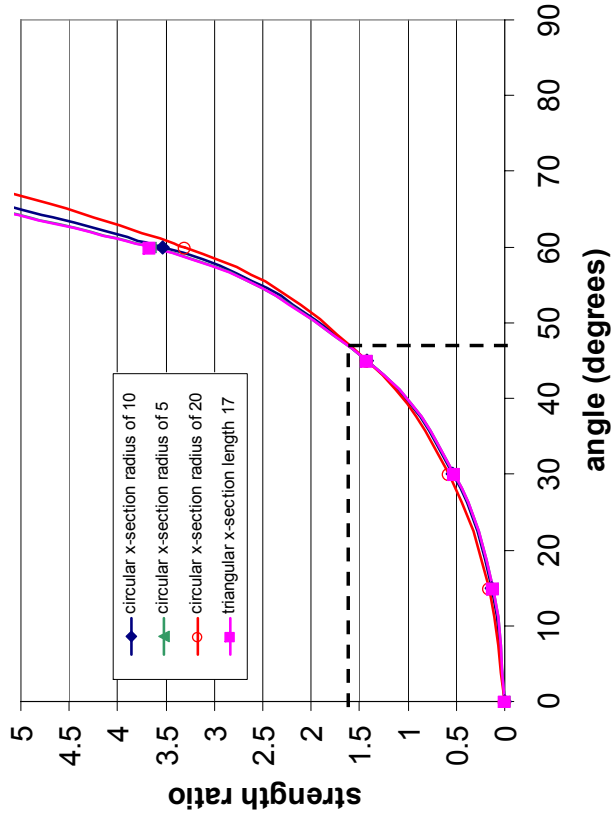


Ratio of strengths in z- and y-directions is

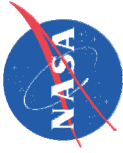
$$\frac{\sigma_z}{\sigma_y} = \frac{4L_2 \sin \theta [2I \cos \theta + L_2 r A \sin \theta]}{[2I \sin \theta + L_2 r A \cos \theta] [2L_2 \cos \theta + \sqrt{2}L_4]}$$

From experimental tensile test specimen the measured ratio of the rise to in-plane strengths is approximately 1.6.

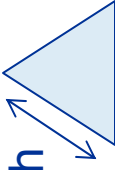
strength ratio versus inclination angle

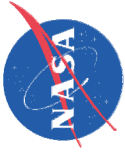


Results indicate that the inclination angle is 47°



Substituting L_2 , L_4 , and the inclination angle into the previous equations for foam modulus and foam strengths and solving for the polymer (strut) modulus and polymer (strut) strengths, we get

		Polymer (Strut) Modulus, MPa (ksi) Using $\theta = 55^\circ$	Polymer (Strut) Strength, MPa (ksi) Using $\theta = 47^\circ$
Circular x-section	$R=5\ \mu\text{m}$	21,500 (3,120)	603 (87.5)
	$R=10\ \mu\text{m}$	1,450 (210)	76 (11)
	$R=20\ \mu\text{m}$	111 (16)	10 (1.5)
Triangular x-section 	$h=17\ \mu\text{m}$	7,300 (1,059)	393 (57)



Summary

- SOFI microstructure were successfully modeled as elongated Tetraikaidechaedron closed cell
- Elastic modulus and Poisson's ratios were estimated from the unit cell model
- The estimation of the polymer solid strength and stiffness was attempted by FEA and Frame models.
- Frame model yields similar results as the finite element model if a circular cross-section with a radius of 10 mm is used.

Future Work

- Refine the FEA model (Nonlinearity) to improve on the failure stress estimates
- Review the modeling of a unit cell in vacuum